

HAIRFOILS: PASSIVE ACTUATORS FOR FLOW CONTROL

A. Bottaro, J. Favier (DICAT, Genova)

&

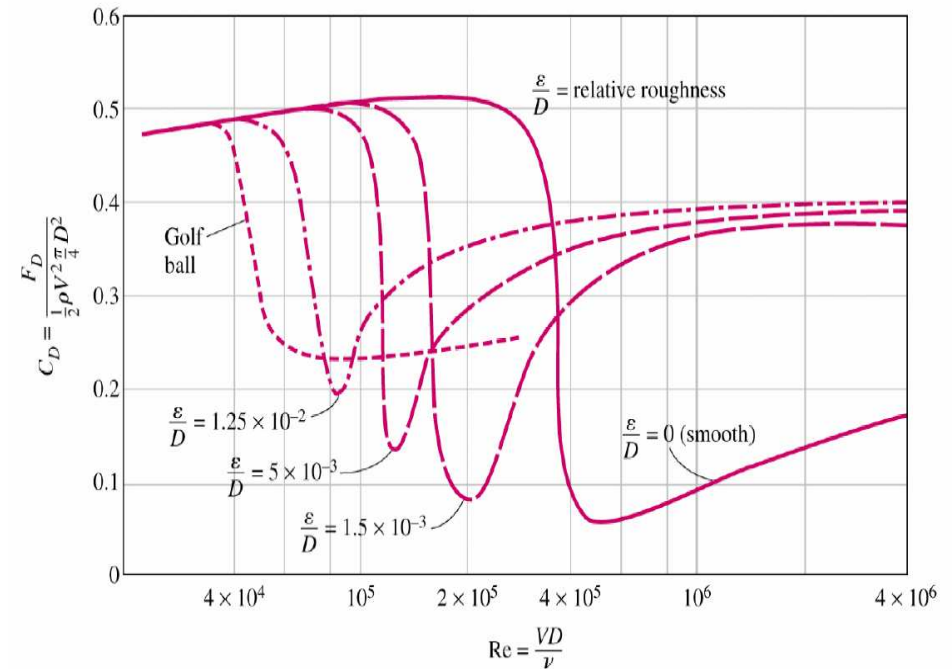
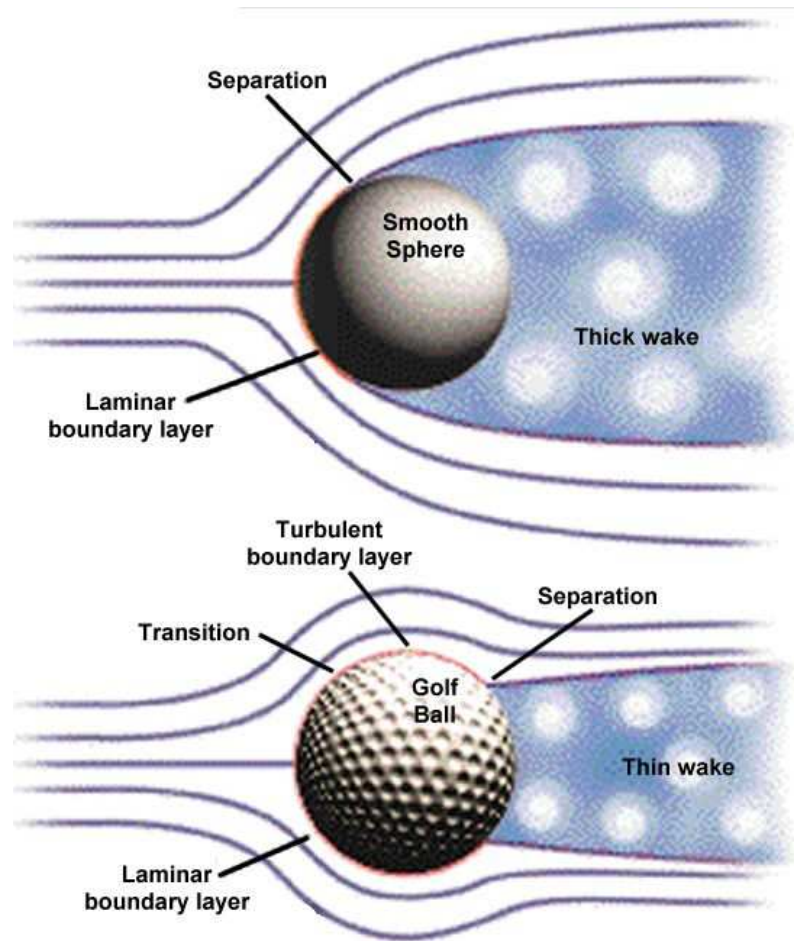
A. Dauplain (CERFACS, Toulouse)

Fluid & Elasticity, Carry-le-Rouet, 23-26 june 2009



How can we reduce pressure drag behind a solid bluff body by a **passive** technique?

How can we reduce pressure drag behind a solid bluff body by a **passive** technique?



Known techniques of passive/active flow control:

- **Injection of micro-bubbles and/or polymers**
- **Riblets**
- **Compliant walls**
- **Viscosity modifier**
- **Vortex generators**
- **...**

Known techniques of passive and/or active flow control:

- Injection of micro-bubbles and/or polymers
- Riblets
- Compliant walls
- Viscosity modifier
- Vortex generators
- ...

Why not use a

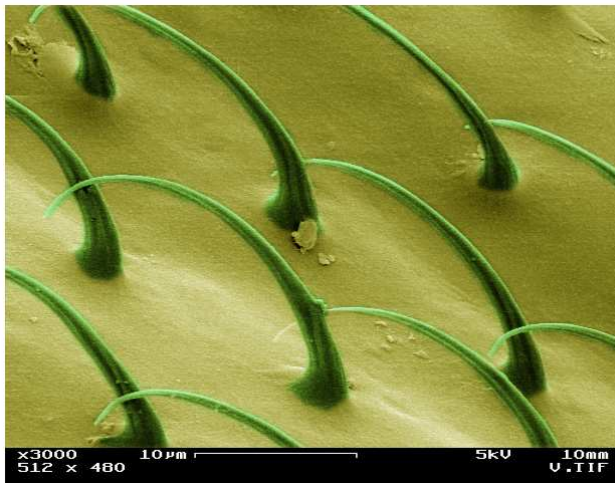
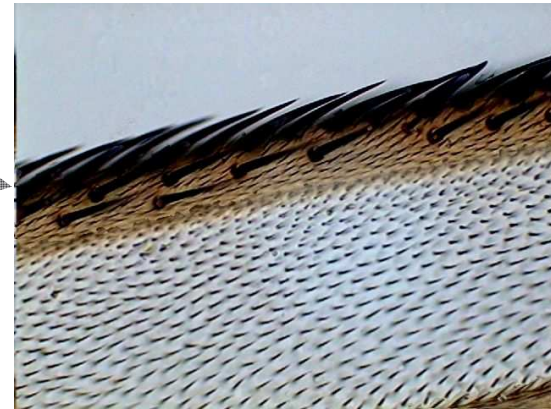
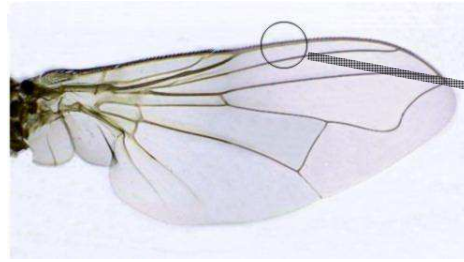
passive hairy coating?



sea otter

How can we increase lift over a streamlined body at incidence by a **passive technique?**

How can we increase lift over a streamlined body at incidence by a **passive** technique?



How can we increase lift over a streamlined body at incidence by a **passive** technique?

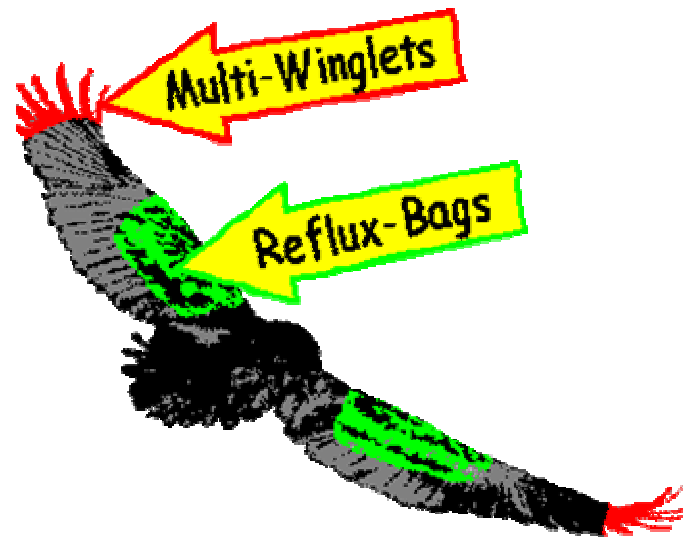


How can we increase lift over a streamlined body at incidence by a **passive** technique?



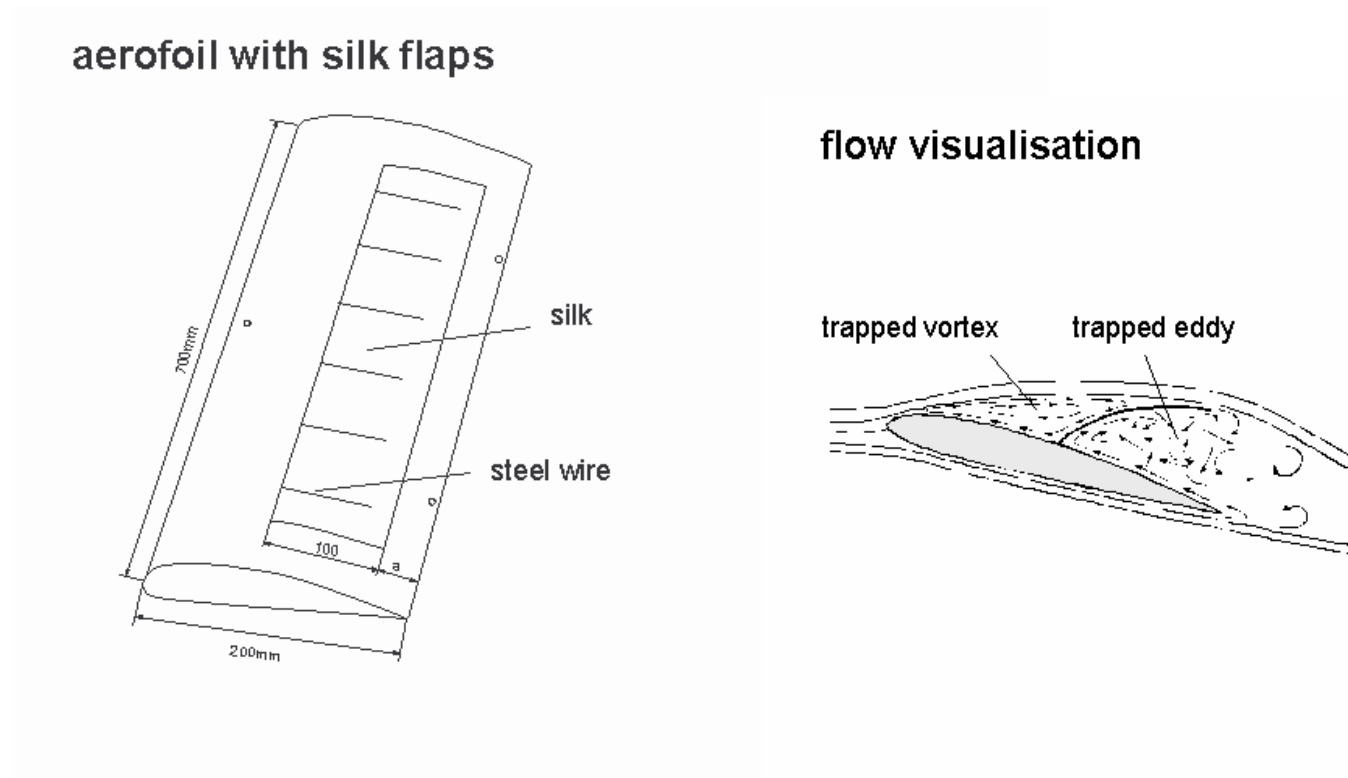
Coverts have also a
aerodynamic role ...

How can we increase lift over a streamlined body at incidence by a **passive** technique?



Prof. Ingo Rechenberg, TU Berlin
<http://www.bionik.tu-berlin.de/institut/xs2vogel.html>

How can we increase lift over a streamlined body at incidence by a **passive** technique?

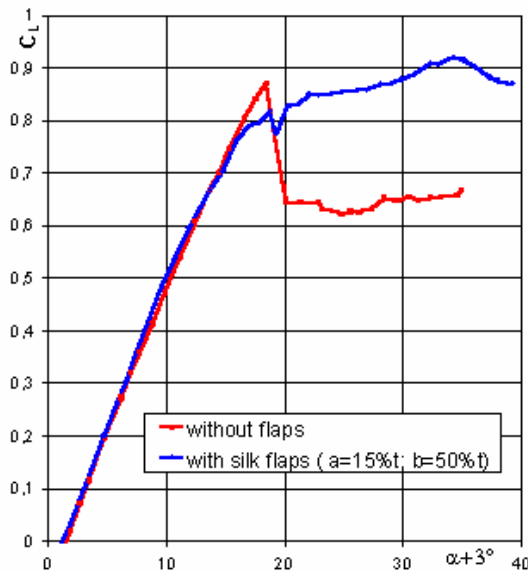


Prof. Ingo Rechenberg, TU Berlin

<http://www.bionik.tu-berlin.de/institut/xs2vogel.html>

How can we increase lift over a streamlined body at incidence by a **passive** technique?

silk flaps



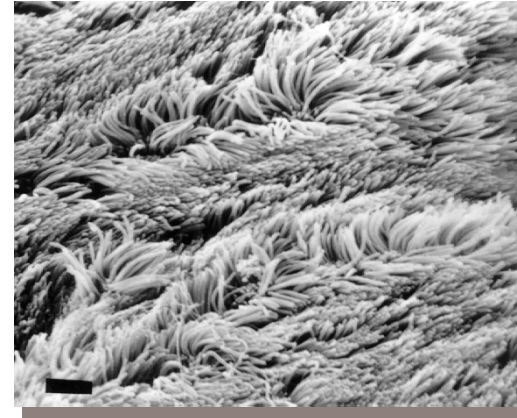
**Flexible, porous flaps
delay stall ...**

Prof. Ingo Rechenberg, TU Berlin

<http://www.bionik.tu-berlin.de/institut/xs2vogel.html>

GOAL: instead of a single flexible flap, let's model of a continuous *hairy/feathery* coating to affect lift and drag

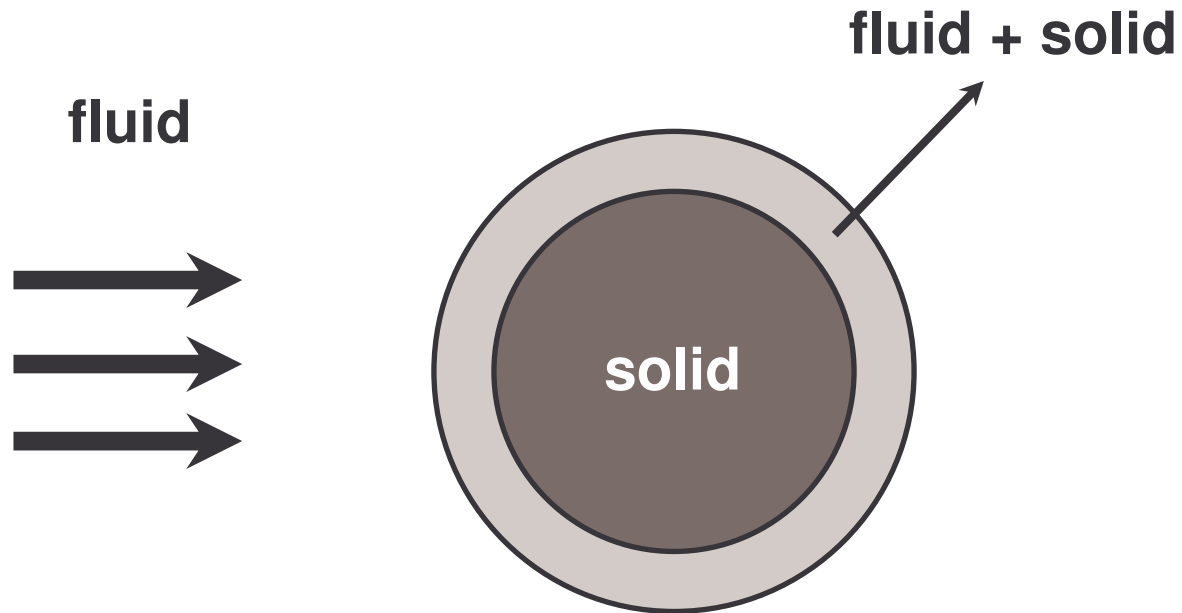
Numerical challenges



- **Model** mechanical properties of **biological surfaces**
- Structures with **large displacements** and **large rotations**
- Interaction between **multiple structures**

Coupling between a layer of oscillating densely packed structures and a unsteady separated boundary layer

The initial configuration



Circular cylinder, $Re=200$

Model of the layer?

Porous, anisotropic and compliant

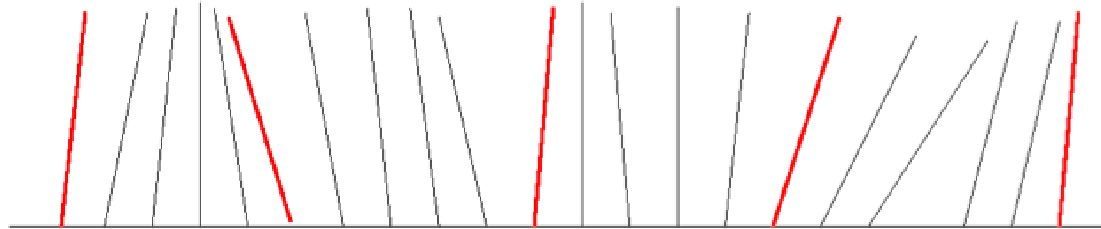
Modeling in 3 points

Modeling all feathers: too heavy ...

→ Must reduce the numbers of degrees of freedom

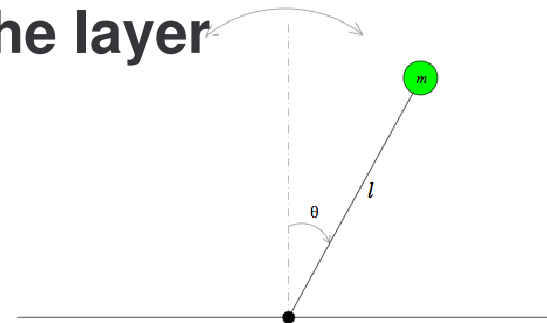
1. **Homogenized approach:** description of the layer in terms of **density** and **direction** of feathers.
2. Motion of the layer reduces to the oscillation of a small number of **reference elements**
3. The fluid “sees” the structures in terms of **volume forces** (and the same for the structures)

Homogenized approach



Dynamics of the layer

Approximation :
Rigid reference element



E. De Langre, ARFM, 2008



Fluid part...

$$\frac{\partial \mathbf{U}}{\partial t} + \nabla(\mathbf{U}\mathbf{U}) = -\frac{\nabla p}{\rho} + \nu \nabla^2 \mathbf{U} + \mathbf{F}; \nabla \cdot \mathbf{U} = 0$$

2D incompressible
Volume forces formulation

Staggered grid
Periodic boundary conditions,
with buffer domain to treat I/O

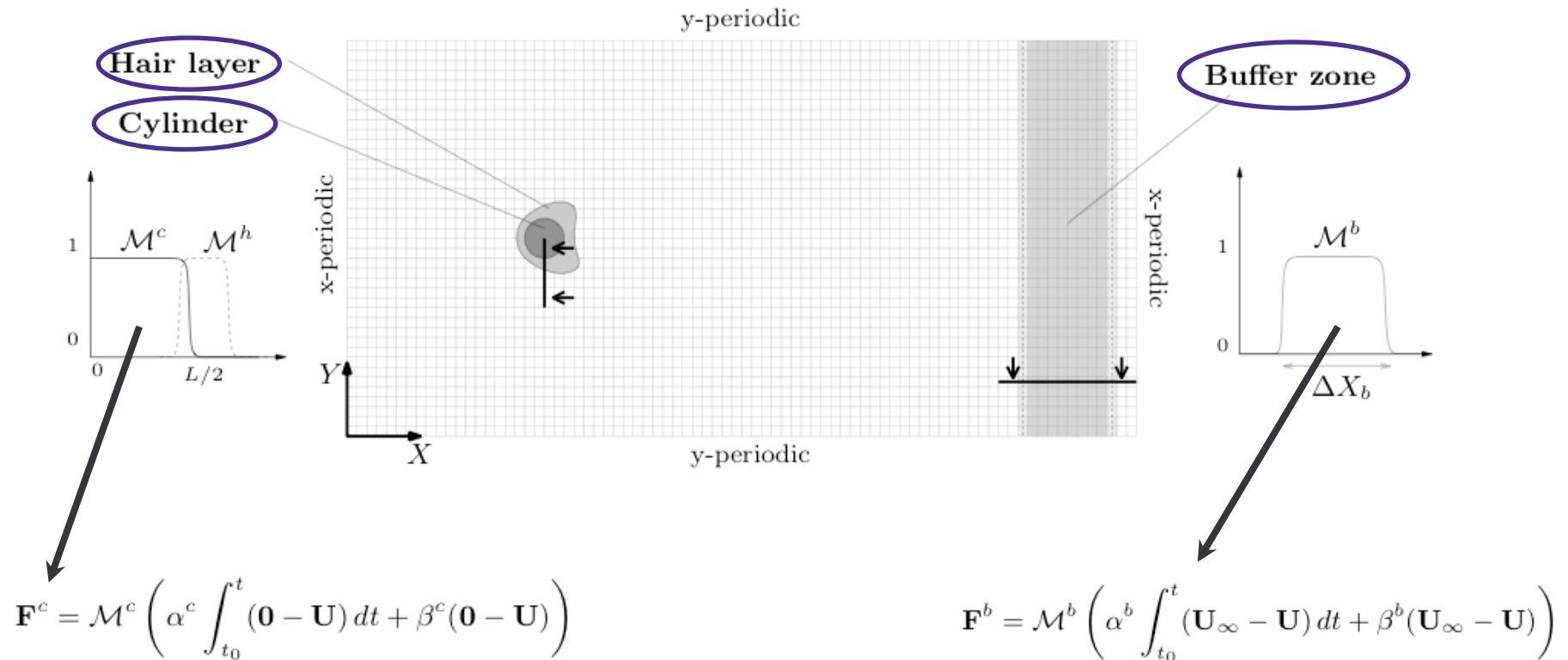
Convective part: Adams-Bashfort
Viscous part: Crank-Nicolson
**Poisson and implicit parts
solved using** conjugate gradient

→ order 2 in space and time

Fluid part ...

$$\frac{\partial \mathbf{U}}{\partial t} + \nabla(\mathbf{U}\mathbf{U}) = -\frac{\nabla p}{\rho} + \nu \nabla^2 \mathbf{U} + \mathbf{F}; \quad \nabla \cdot \mathbf{U} = 0$$

Regular cartesian mesh 200 x 400 (10L x 20L)



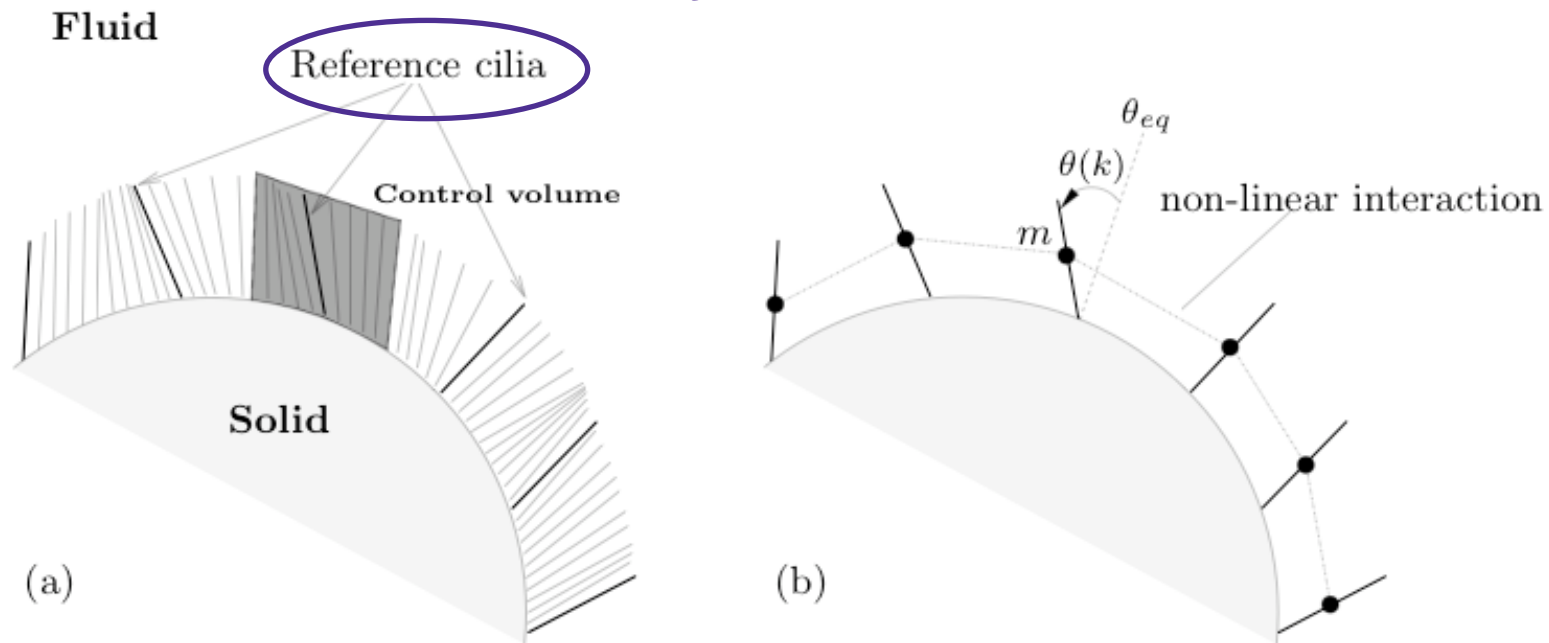
$$\mathbf{F}^c = \mathcal{M}^c \left(\alpha^c \int_{t_0}^t (\mathbf{0} - \mathbf{U}) dt + \beta^c (\mathbf{0} - \mathbf{U}) \right)$$

$$\mathbf{F}^b = \mathcal{M}^b \left(\alpha^b \int_{t_0}^t (\mathbf{U}_\infty - \mathbf{U}) dt + \beta^b (\mathbf{U}_\infty - \mathbf{U}) \right)$$

$$\mathbf{F} = \mathbf{F}^c + \mathbf{F}^b + \mathbf{F}^h$$

Structure part ...

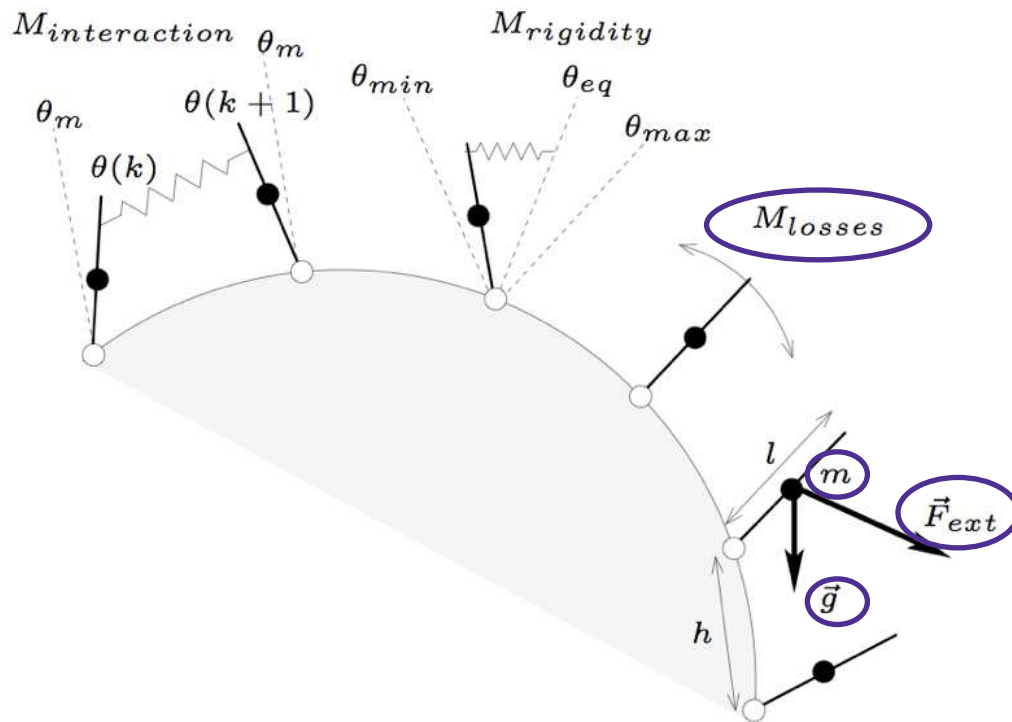
“skeleton” of
the layer



The dynamics of the layer is governed by
the reference elements

Structure part ...

→ The “skeleton” of the layer is governed by **six terms** in the angular momentum equation for each element



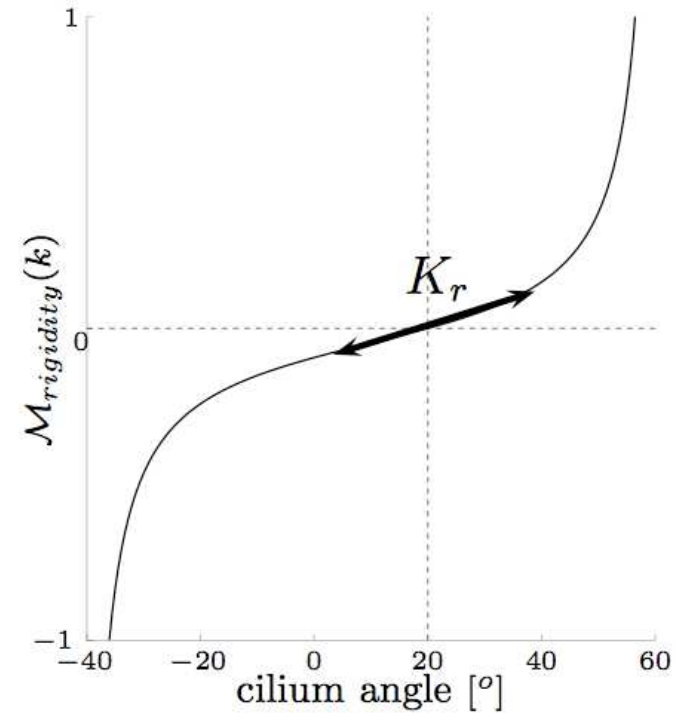
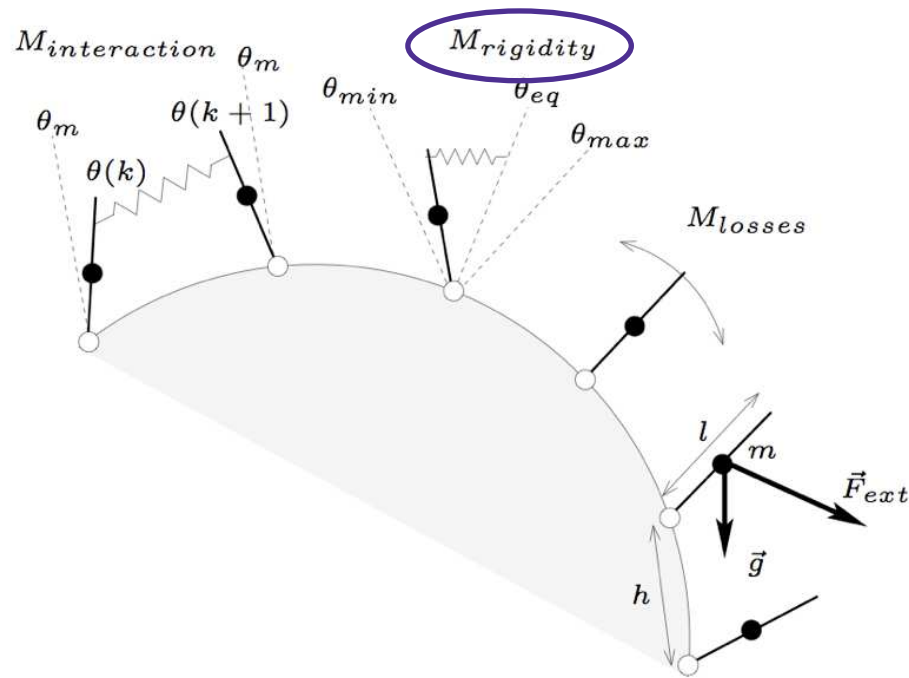
$$M_{losses}(k) = -K_l \dot{\theta}(k)$$

$$M_{inertia}(k) = -m \frac{l^2}{4} \ddot{\theta}(k)$$

$$M_{gravity} = mg \frac{l}{2} \sin(\theta)$$

$$M_{ext}(k) = \frac{l}{2} F_n(k)$$

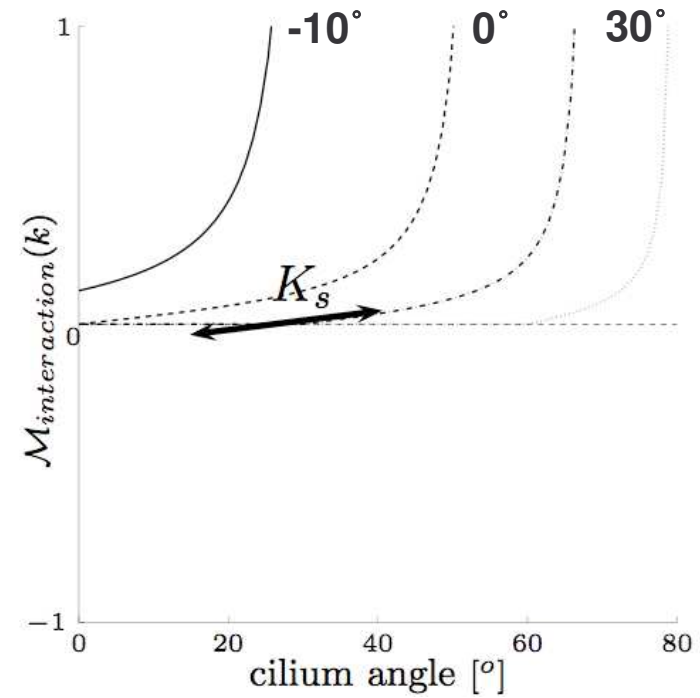
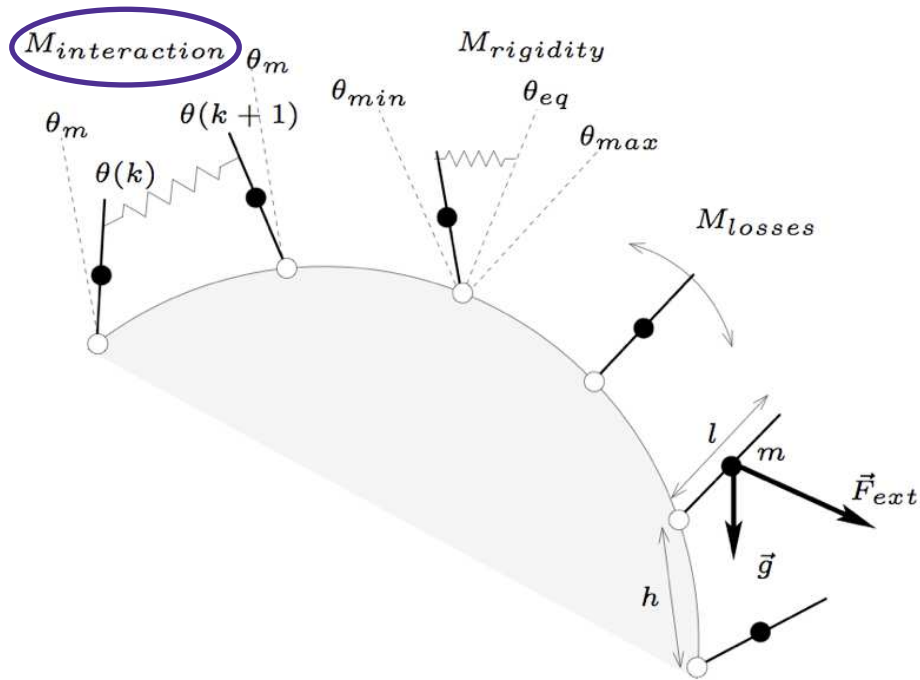
Structure part ...



$$M_{rigidity}(k) = -K_r \frac{P(\theta) - P(\theta_{eq})}{P'(\theta_{eq})}$$

avec $P = \tan(a\theta + b)$ $a = \pi / (\theta_{max} - \theta_{min})$
 $b = -a(\theta_{max} + \theta_{min}) / 2$

Structure part ...



$$M_{interaction}(k) = K_s \tanh \frac{2l \sin(\theta(k) - \theta_m)}{h \cos(\theta_m)}$$

avec $\theta_m = \frac{\theta(k) + \theta(k+1)}{2}$

Structure part ...

Explicit resolution: Runge-Kutta 4

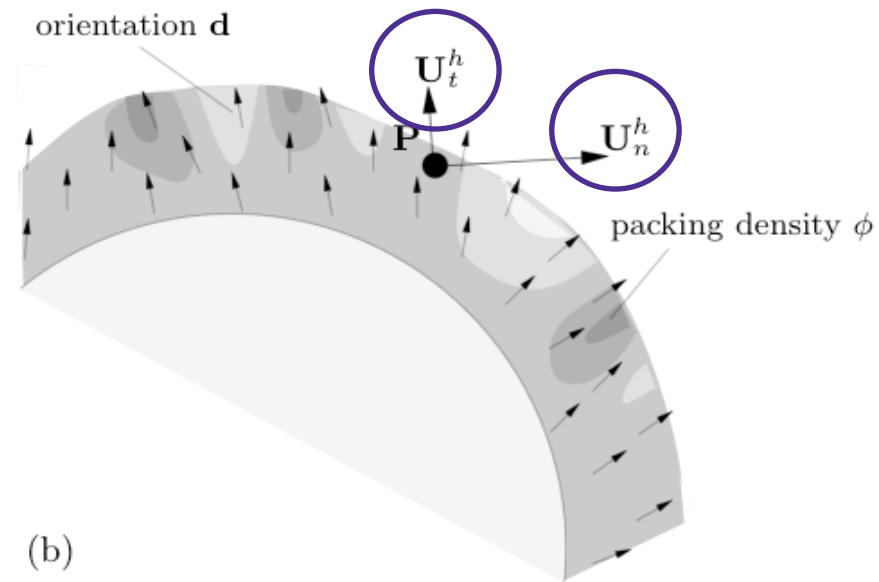
$$m(l/2)^2\ddot{\theta}(k) = M_{spring}(k) + M_{rigidity}(k) + M_{dissip}(k) + M_{inertia}(k) + M_{ext}(k) , k = 1, \dots, N_c$$

Equilibrium is reached after a sufficient number of sub-iterations
For small masses: oscillations at long subtimes

Implicit resolution: Non-linear Conjugate gradient

$$M_{inertia}(k) + M_{spring}(k) + M_{rigidity}(k) + M_{dissip}(k) + M_{inertia}(k) + M_{ext}(k) , k = 1, \dots, N_c$$

How to evaluate the force imposed by the fluid onto the structures ...

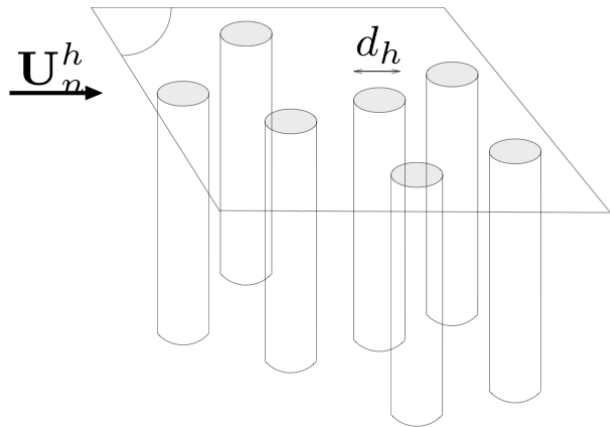


Decomposition of the local relative velocity into a tangential and a normal contribution

Homogenized part (fluid+structure) ...

- Each cilium is a circular cylinder
- At each point along the beam, the force is decomposed into a tangential and a normal contribution
- Force on a random cluster of cylinders

Estimate of F_n



$$\frac{||\mathbf{F}_n^h||}{\mu ||\mathbf{U}_n^h||} = c_0(\phi) + c_1(\phi) Re_n^h$$

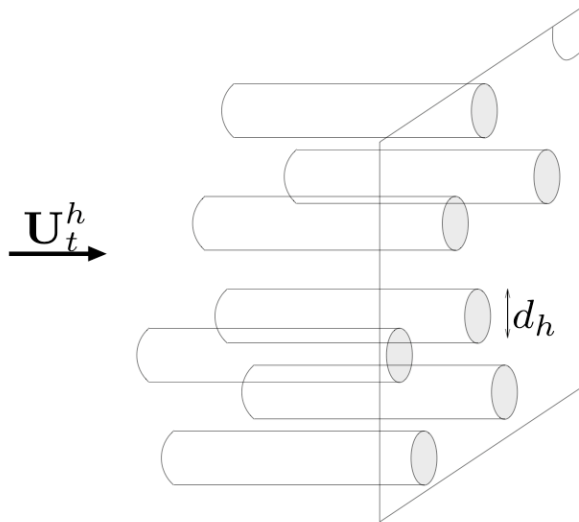
theoretical (Re=0) empirical Re<180

Koch & Ladd, JFM 1997

$$||\mathbf{F}_n^h|| = f_2(\phi, Re_n^h)$$

Homogenized part (fluid+structure) ...

Estimate of F_t



→ For $Re = 0$: Stokes approximation:

$$\frac{\mathcal{F}_t^h}{\mu \|\mathbf{U}_t^h\|} = \frac{8\pi(1 - \phi)^2}{\phi - 1 + \frac{2}{\phi - 1} \ln \phi - 2}$$

$\mathbf{U}_t^h(1 - \phi)$: local velocity through the pores

→ For $Re < 180$: same scaling in Reynolds as for F_n

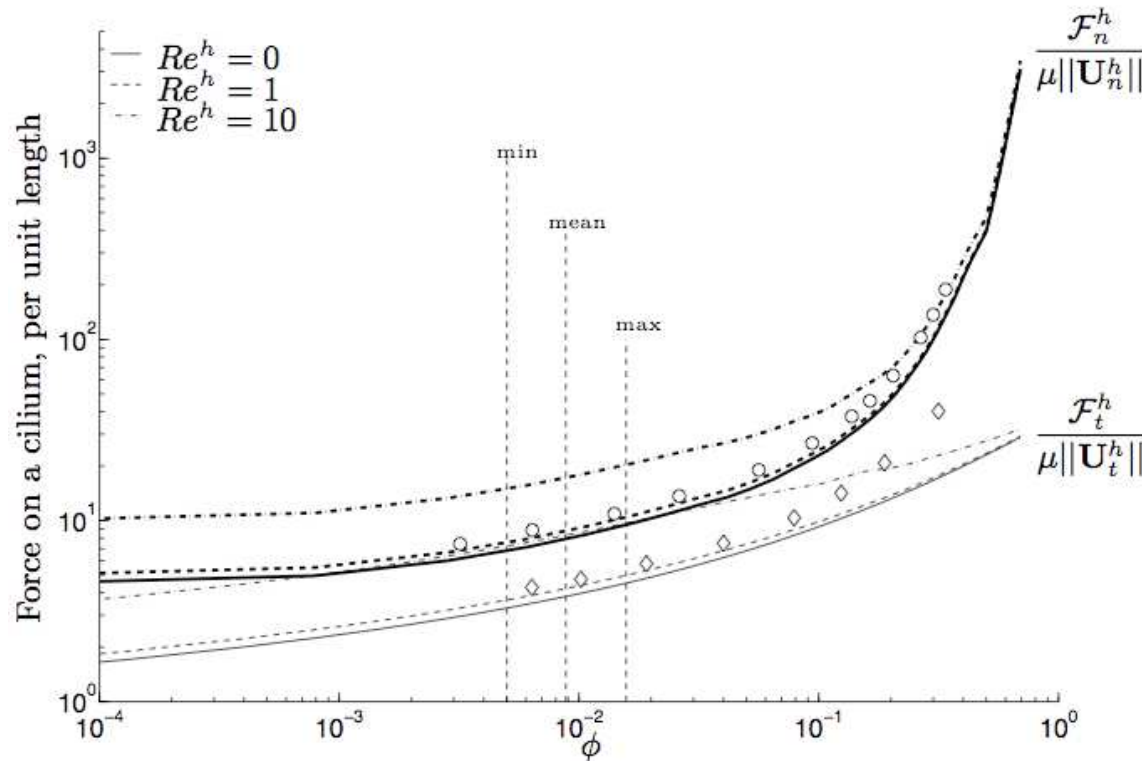
$$\|\mathbf{F}_t^h\| = f_1(\phi, Re_t^h)$$

Homogenized part (fluid+structure) ...

Inner constants of the layer:

Density (nb/cm²),

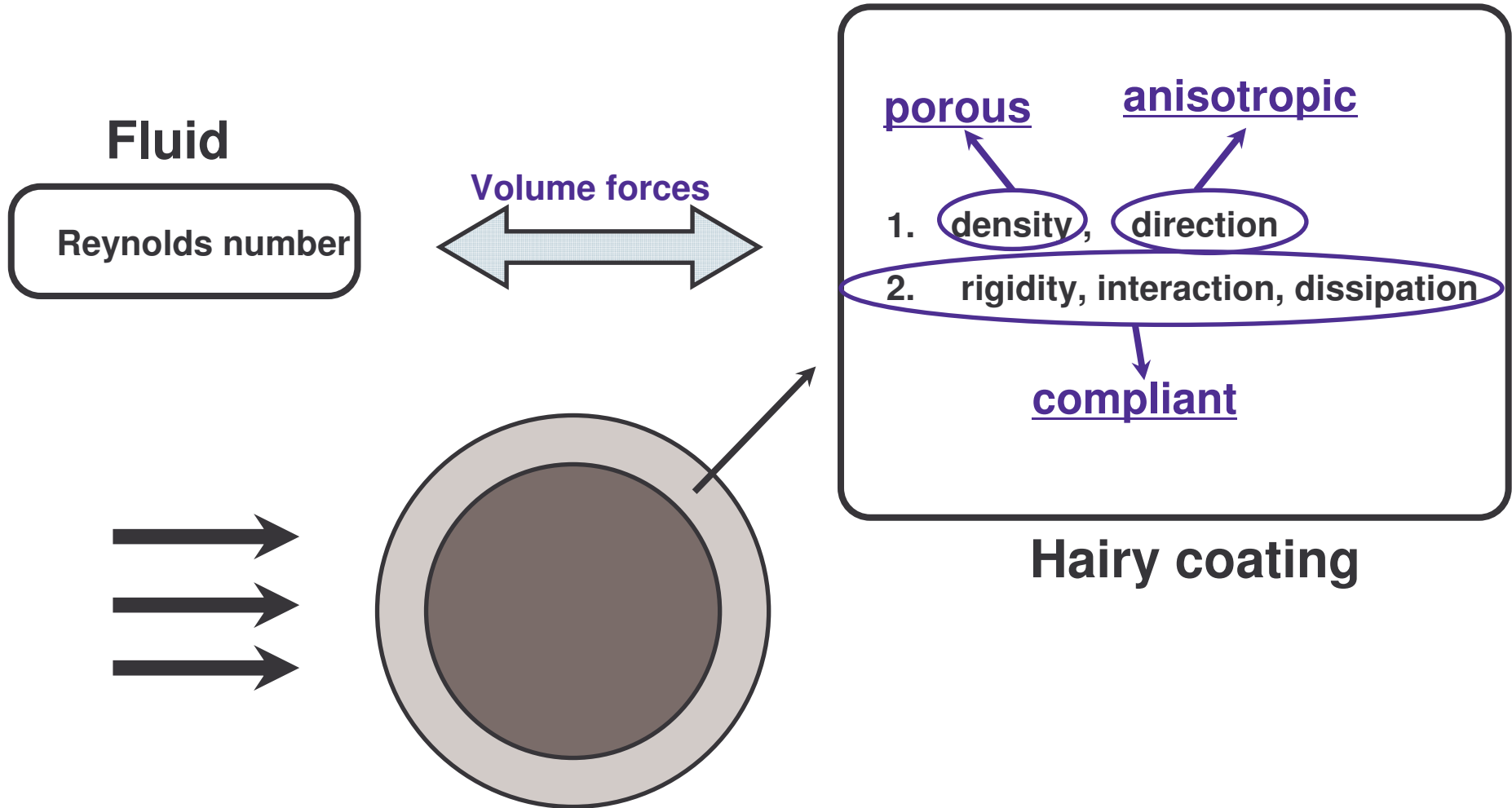
Diameter of cilia



Symbols :
theoretical model by
Howells, JFM 1997

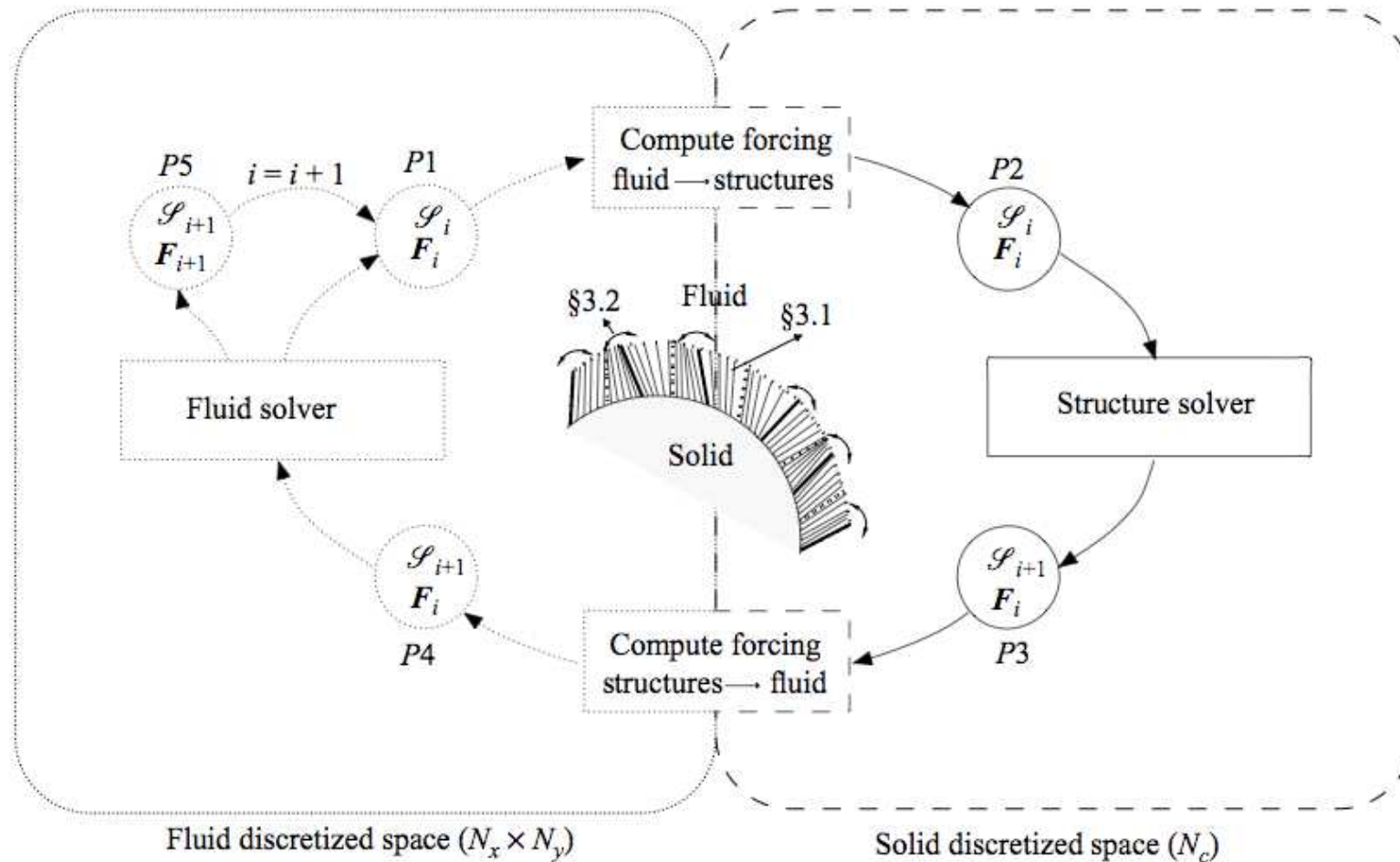
$$\longrightarrow F_{ext}(k) = \int_{V_{control}(k)} \|\mathbf{F}^h\| dV$$

Global overview



Algorithm

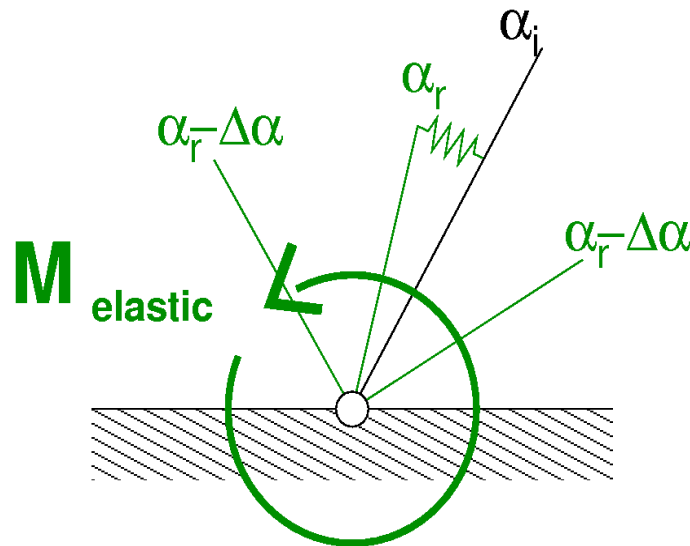
Partitioned staggered in time



All routines in f90

Parallelized (auto and openmp), portable and runs on clusters

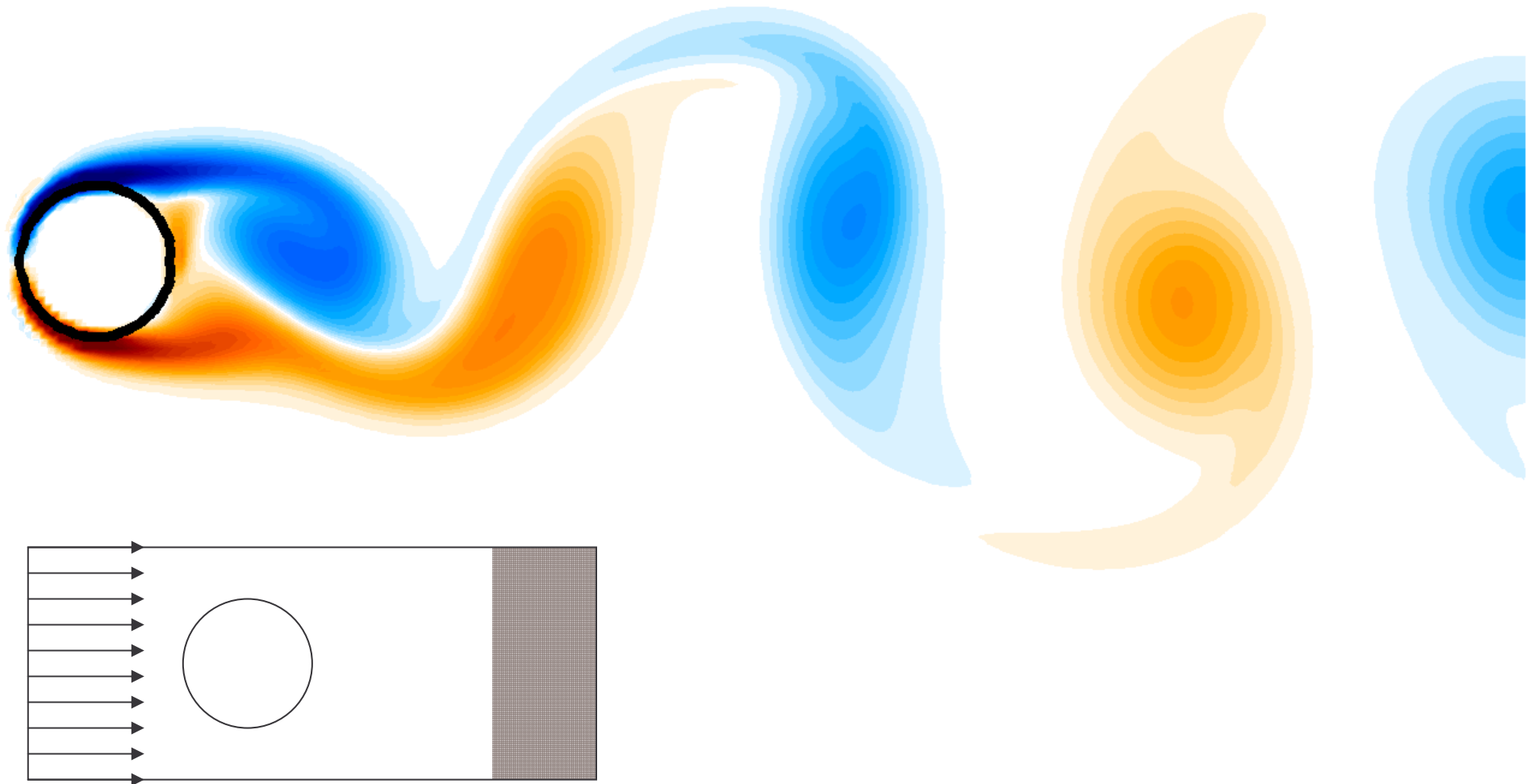
In the structural model, the **rigidity/elastic term**, which models the structural flexibility of the hairy layer, is the most significant. It defines a natural time scale of the layer, through which a coupling with the fluid is allowed



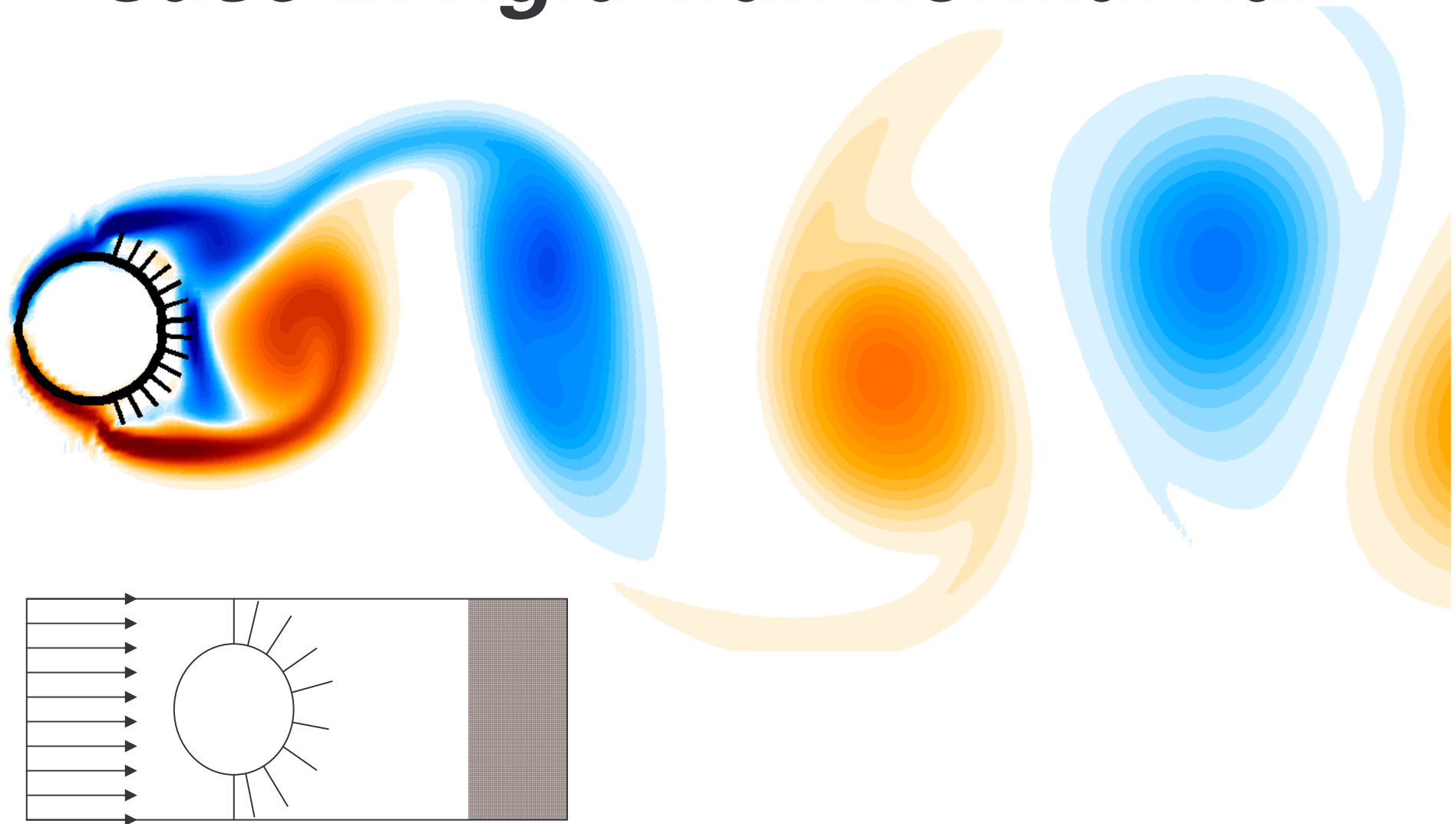
$$T_{\text{structure}} \approx \pi l \sqrt{(m/K_r)}$$

$$T_{\text{fluid}} \approx St^{-1} D/U_{\infty}$$

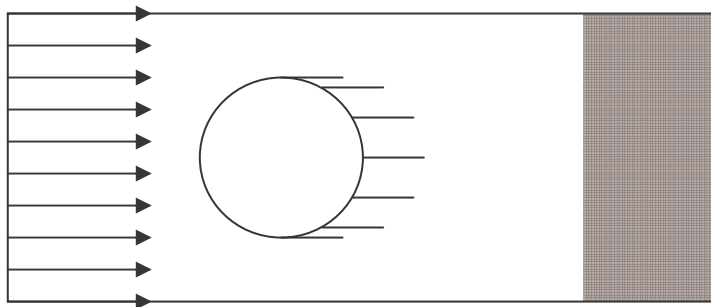
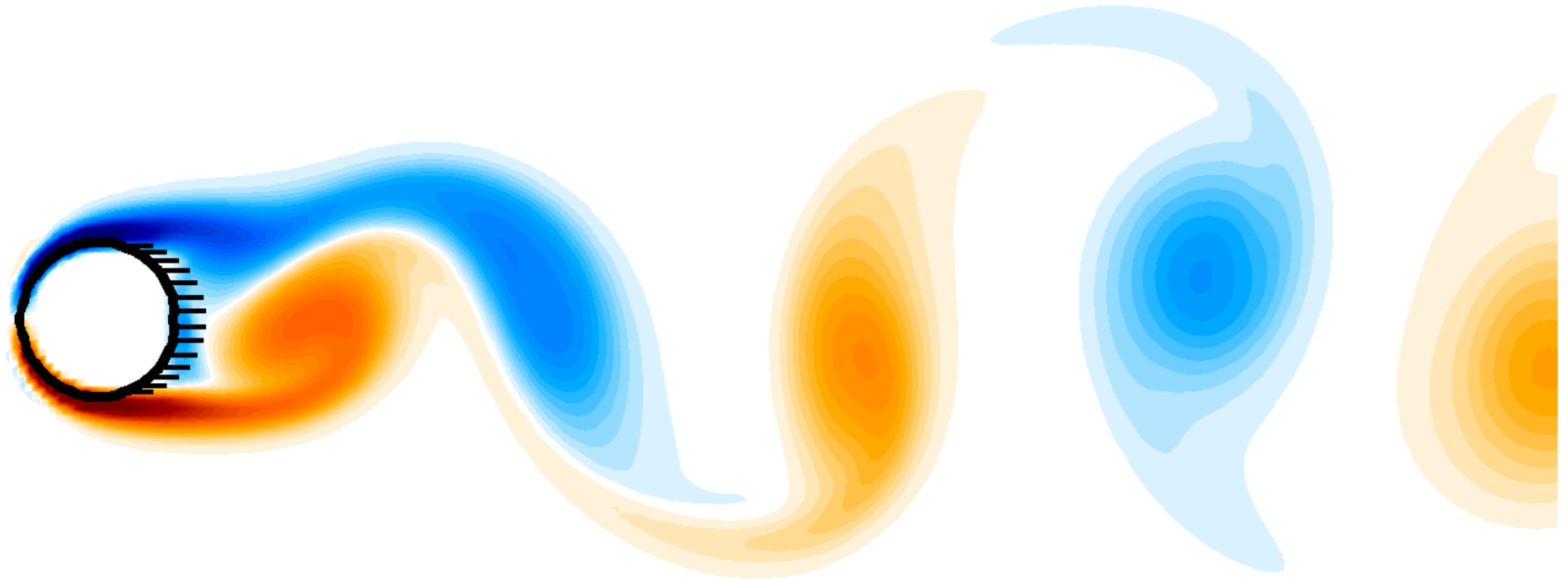
Case 1: bare cylinder



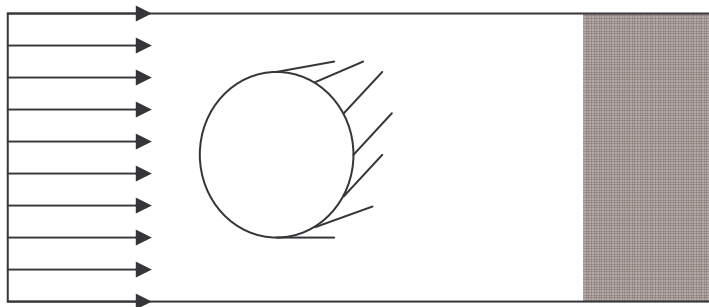
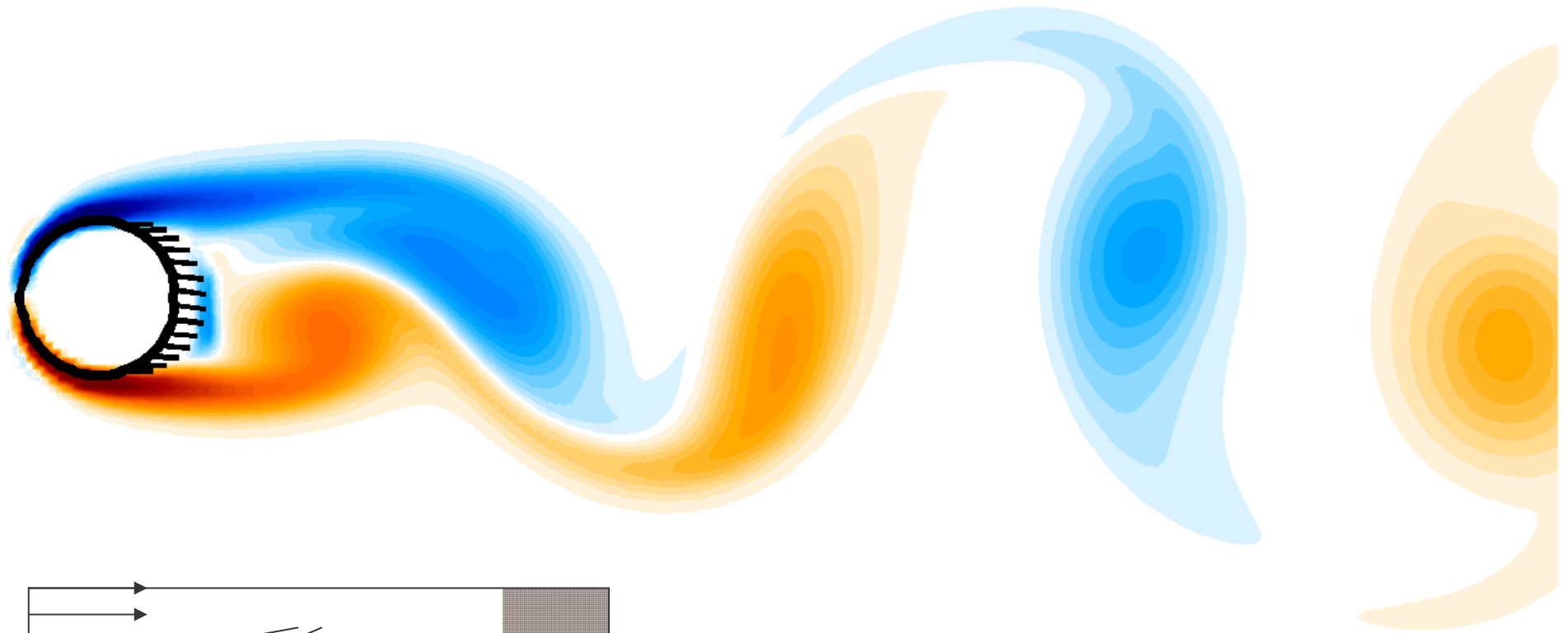
Case 2: rigid wall-normal hair



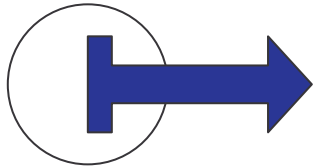
Case 3: rigid longitudinal hair



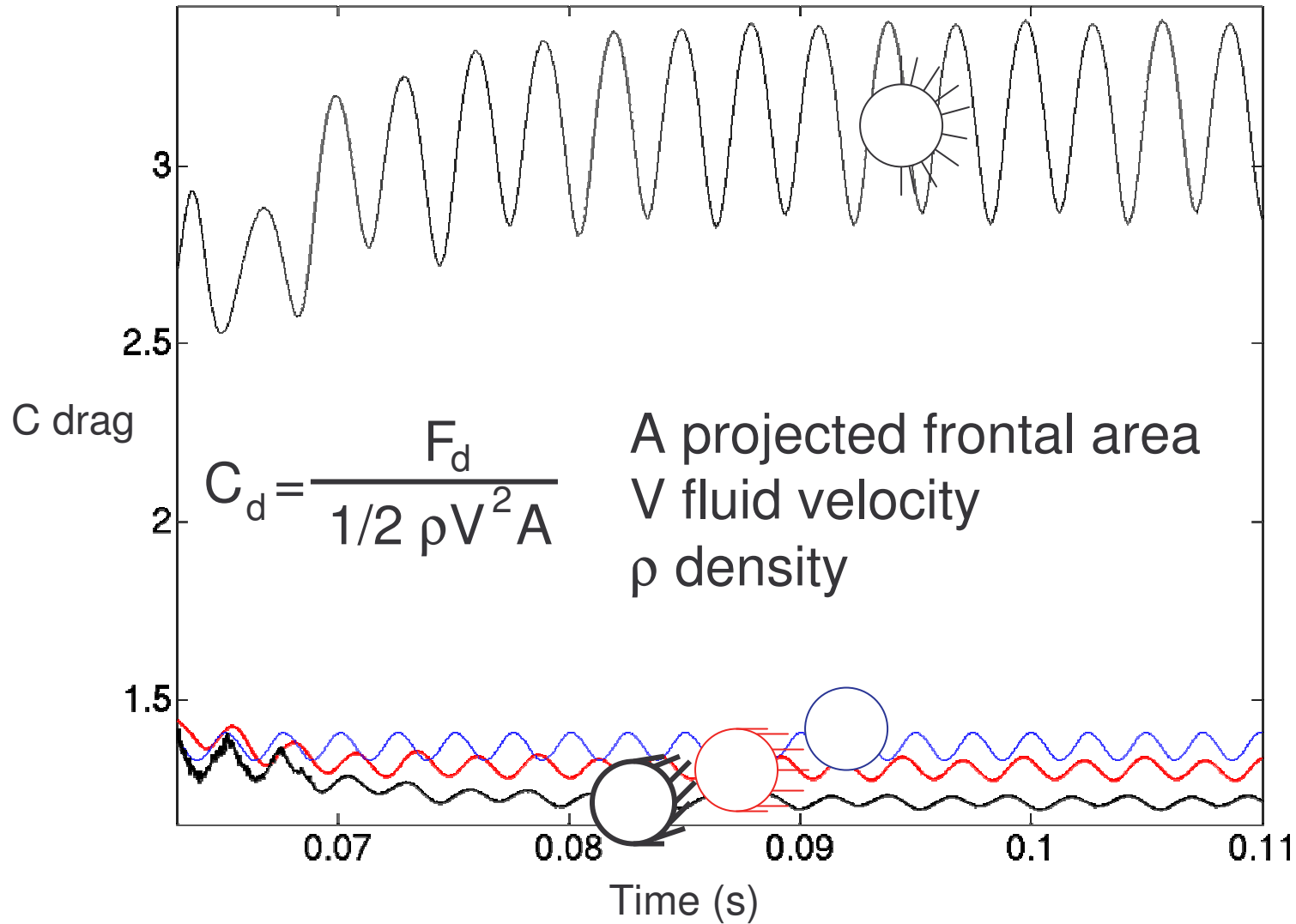
Case 4: moving hair

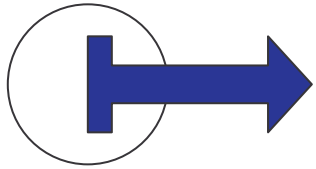


$$T_{fluid} \approx 4 T_{structure}$$

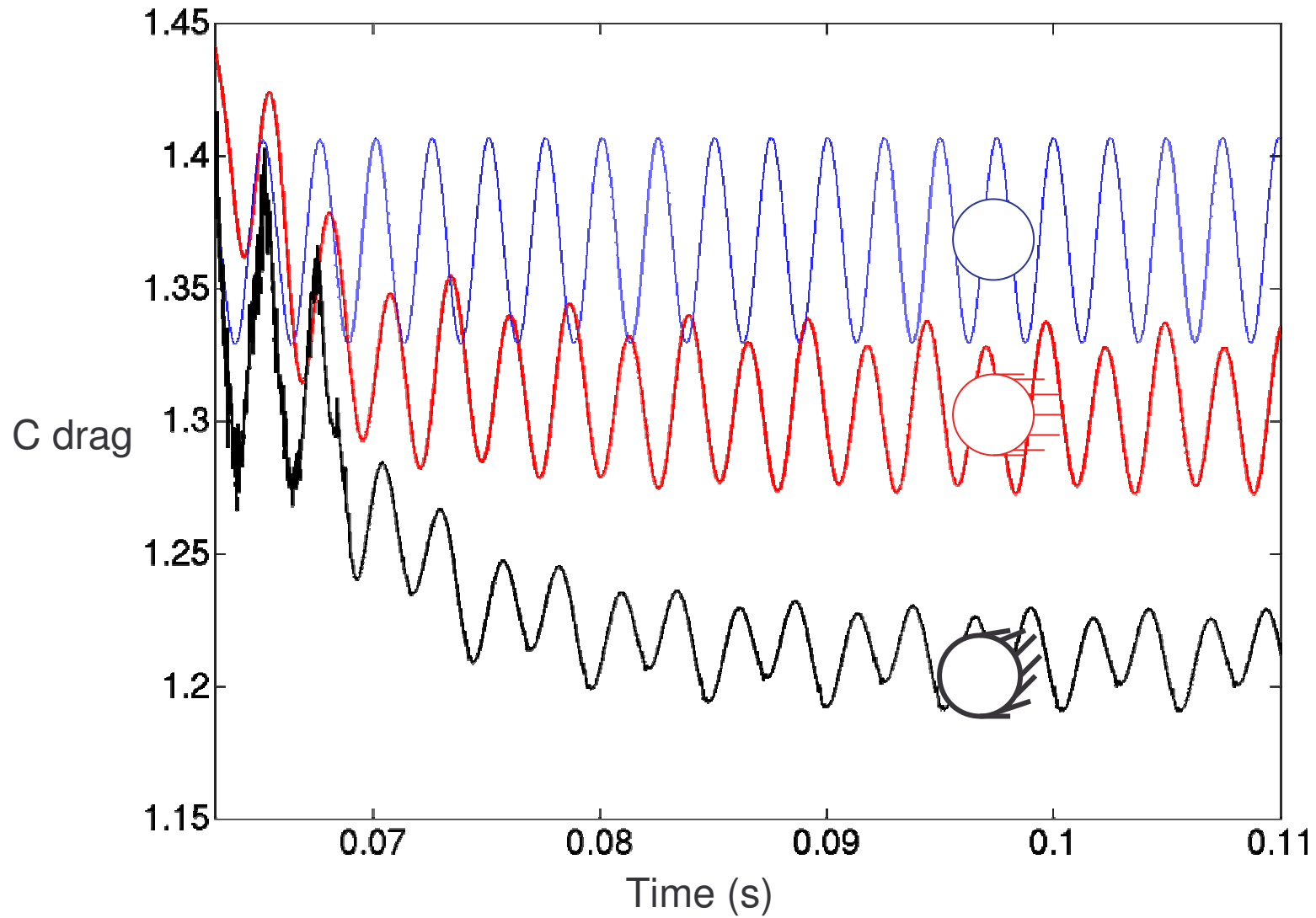


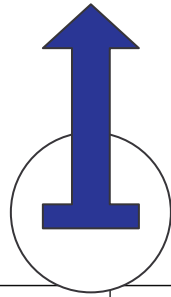
Drag



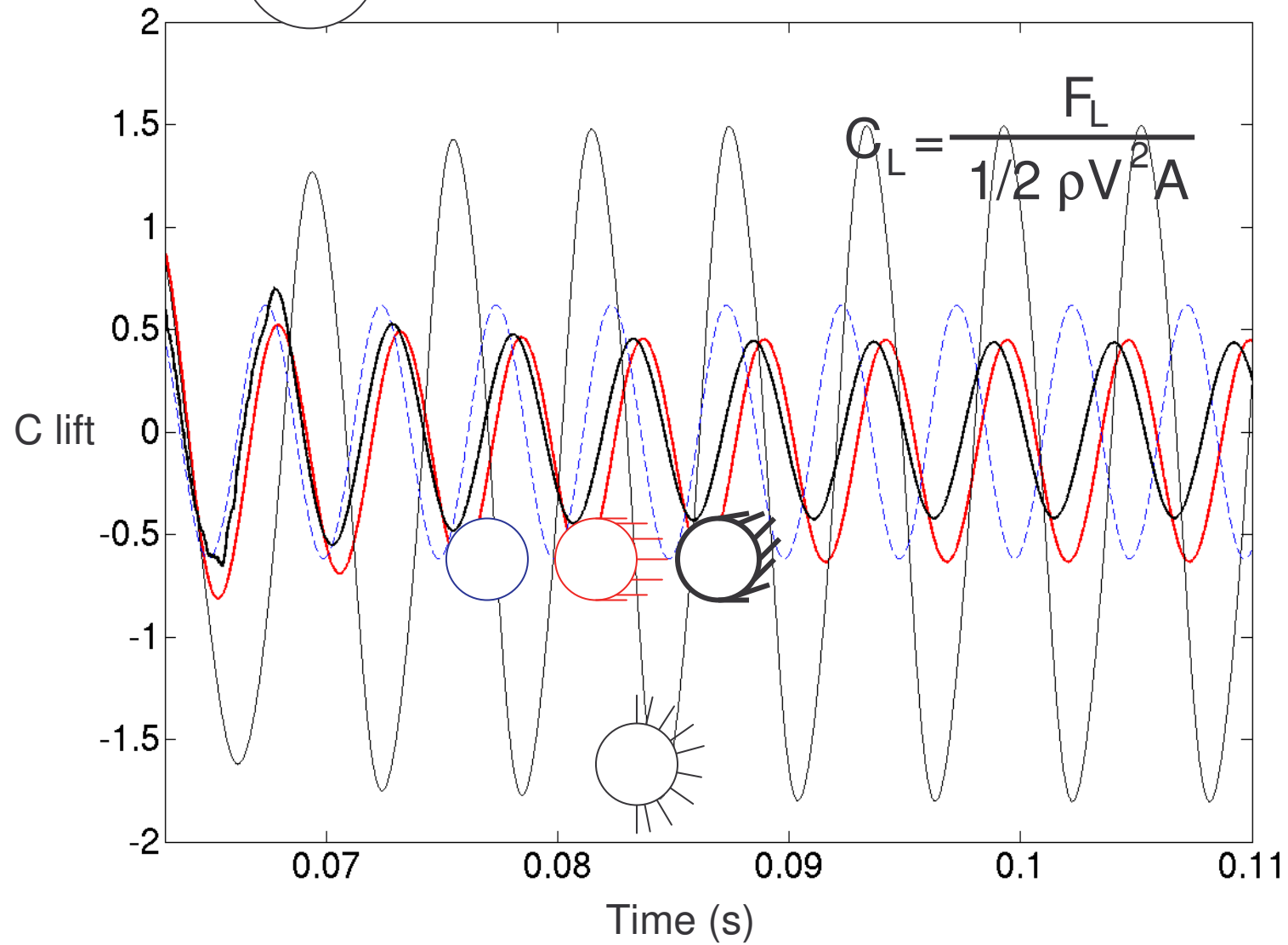


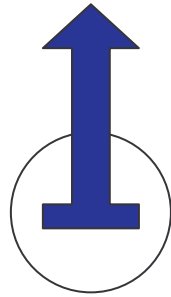
Drag (ctd.)



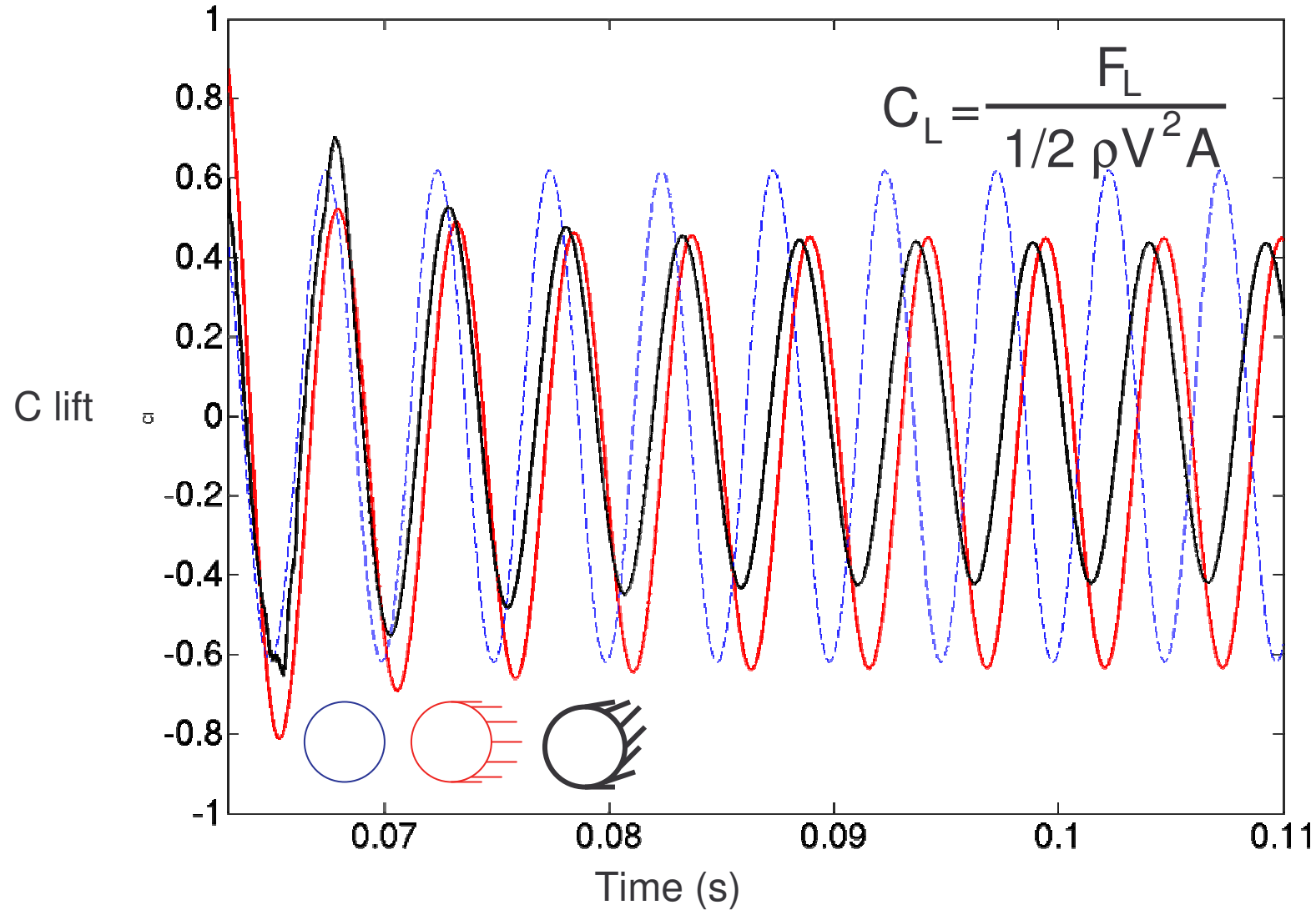


Lift

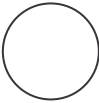
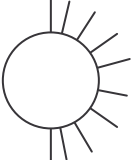
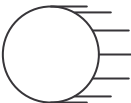





Lift (ctd.)

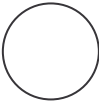
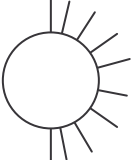
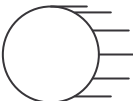



Aerodynamic performances

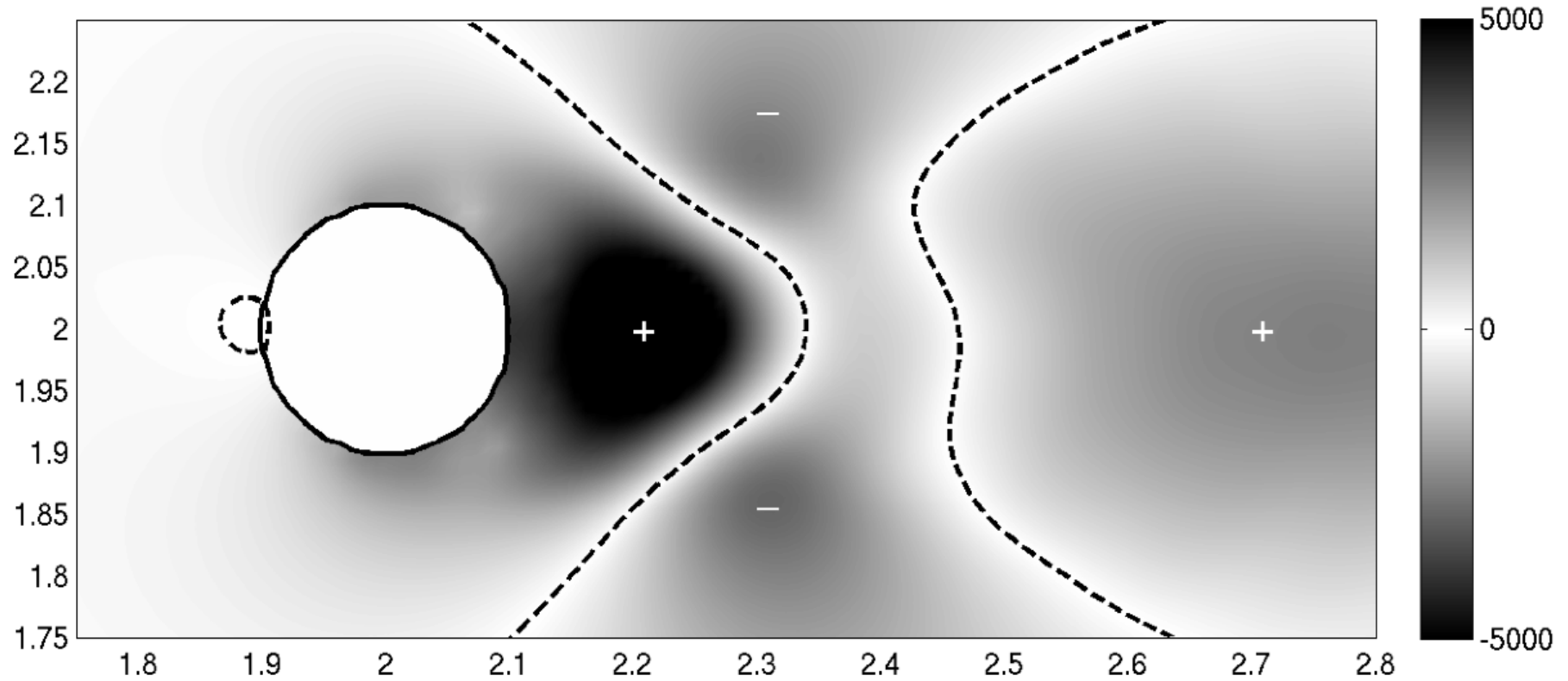
		Cd	Cd'	Cl'	St
Case 1		1.3689 (1.39;1.356)	0.0274	0.4381	0.199 (0.199;0.198)
Case 2		3.1464	0.1943	1.1376	0.1946
Case 3		1.3035	0.0207	0.3839	0.1916
Case 4		1.2109	0.012	0.3008	0.1661

(Bergmann et al. Phys. Fluids 2005 ; He et al J. Fluid Mech. 2000)

Aerodynamic perf.(ctd.)

		Cd	Cd'	Cl'	St
Case 1		ref	ref	ref	ref
Case 2		+130%	+608%	+160%	-2.21%
Case 3		-4.78%	-24.54%	-12.37%	-3.71%
Case 4		-11.54%	-56.09%	-31.34%	-16.53%

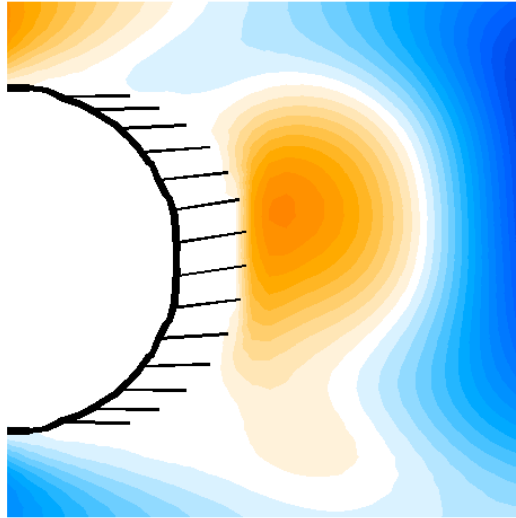
Physical mechanism



Difference of time-averaged pressure field

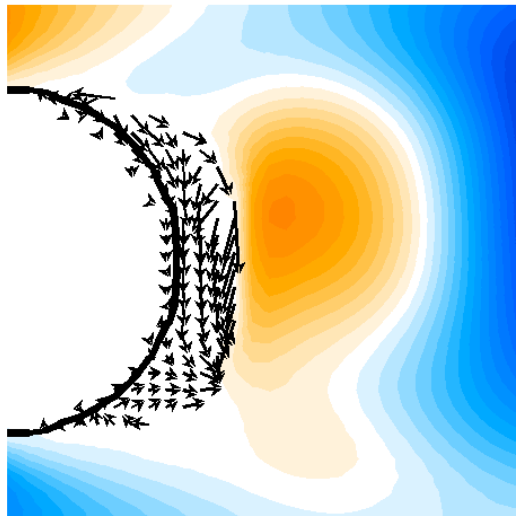
$$\langle P \text{ with hair} \rangle - \langle P \text{ ref} \rangle$$

Physical mechanism



Contours of vertical velocity

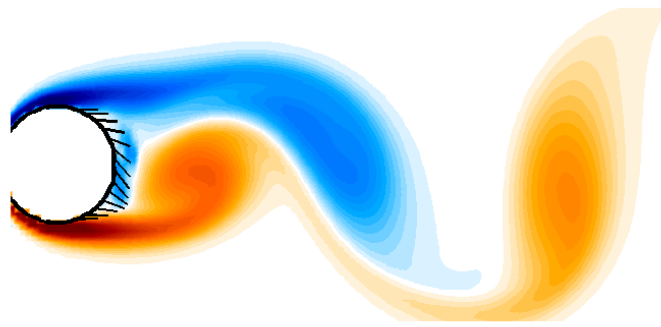
Movements of *reference* cilia



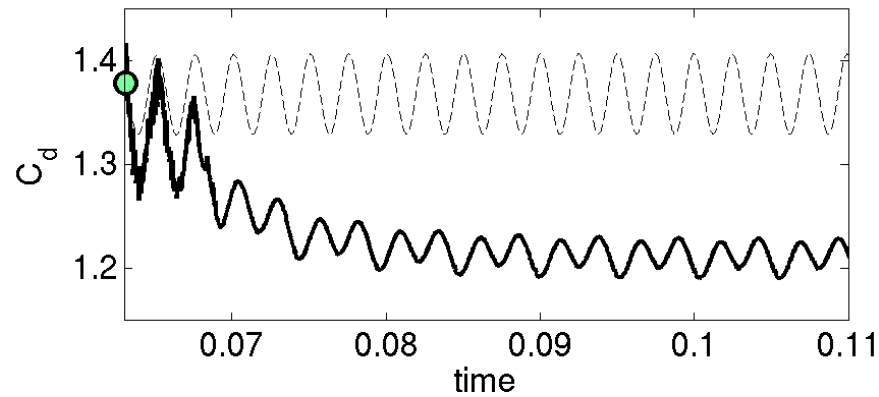
Contours of vertical velocity

Force field

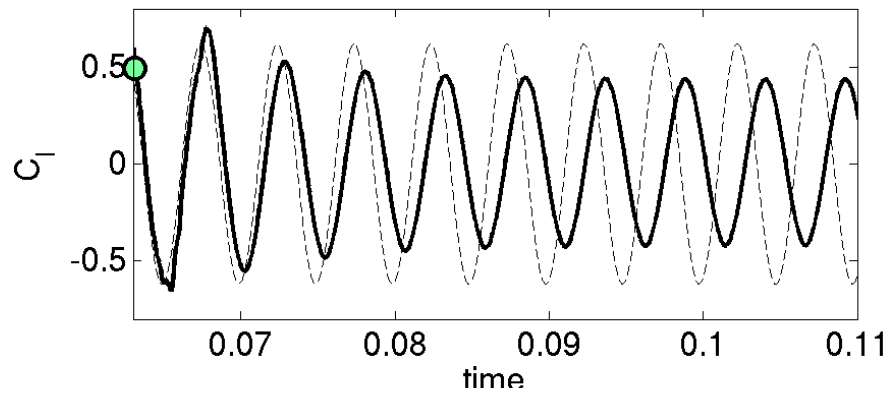
The hairy layer counteracts
flow separation



Optimal self-adaptive hairy layer



15% drag reduction

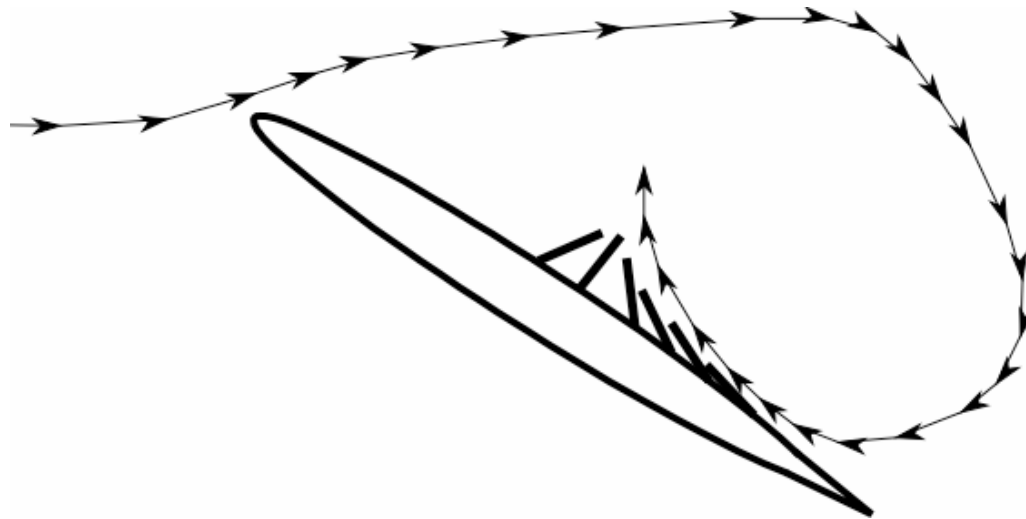


40% reduction in lift fluctuations

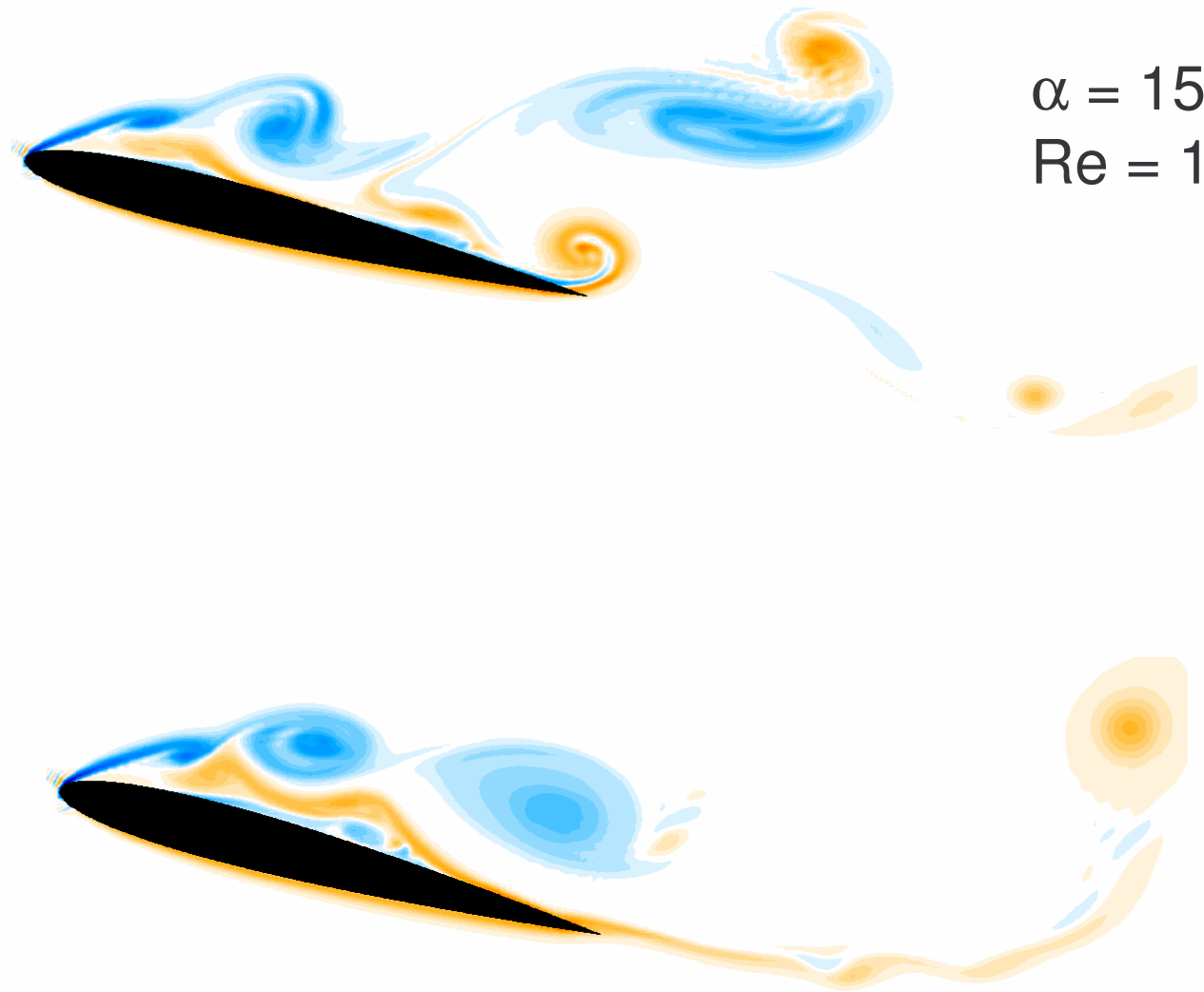
Reducing pressure drag:

- ✓ Simulations show a reduction of pressure drag on a cylinder for a unsteady laminar flow ($Re = 200$).
- ✓ The motion of the hairy structures can improve aerodynamic performances
- ✓ The structural parameters of the actuators have been optimised
- ✓ Immediate perspectives concern flexible rods and turbulent configurations; possible applications to small underwater vehicles and to UAV/MAV (in the aeronautical field)

and now, what about lift?



Consider a hairfoil: the control elements (the *feathers*) must be placed in the position of largest *sensitivity* to achieve an effect



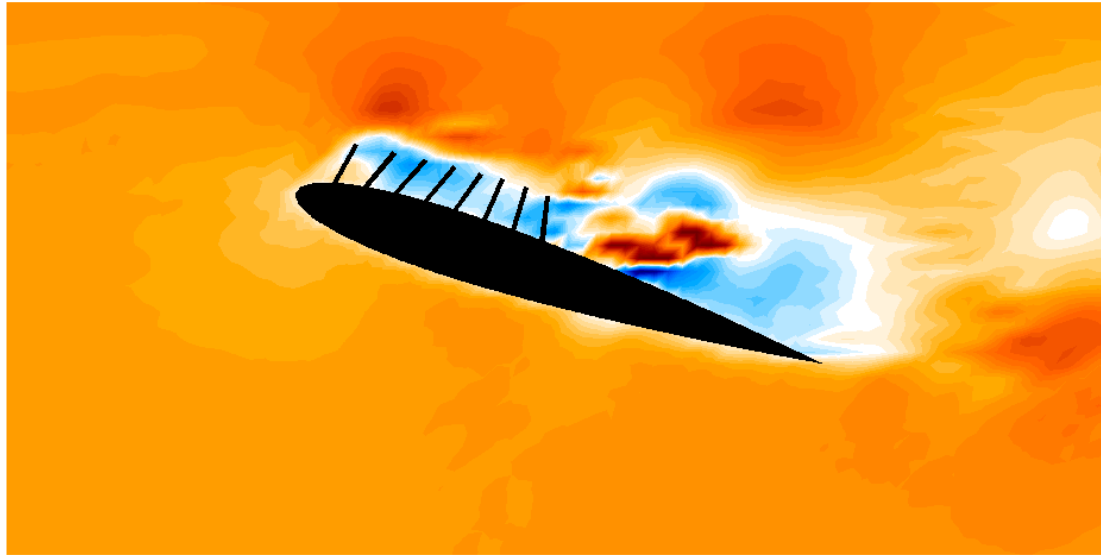
$\alpha = 15^\circ - 18^\circ$
 $Re = 10^4$

Preliminary runs with control elements going from 0.12 chord to 0.95 chord

$$\alpha = 15^\circ \quad \langle C_D \rangle = 0.284 \quad \langle C_L \rangle = 0.579$$

$T_{fluid} = 0.5 T_{structure}$	+ 1.35%	- 13%
$T_{fluid} = T_{structure}$	+ 2 %	- 10%
$T_{fluid} = 2 T_{structure}$	+ 3%	- 9%
$T_{fluid} = 4 T_{structure}$	- 0.2 %	+ 2.5%
$T_{fluid} = 8 T_{structure}$	-7 %	- 11%

Results are similar when $\alpha = 18^\circ$, except that now $\langle C_L \rangle$ increases the most for $T_{fluid} = 2 T_{structure}$



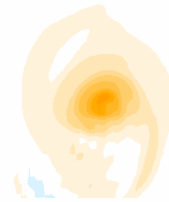
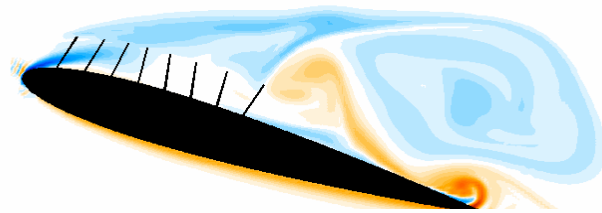
The amplitude of the oscillations decreases
(the system's stability improves) as $T_{structure} \nearrow$

(i.e. $m \nearrow$ $l \nearrow$ $K_r \searrow$)

**A parametric resonance must be triggered
to optimise the response of the system**

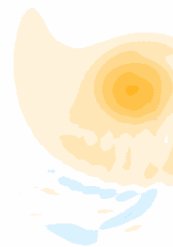
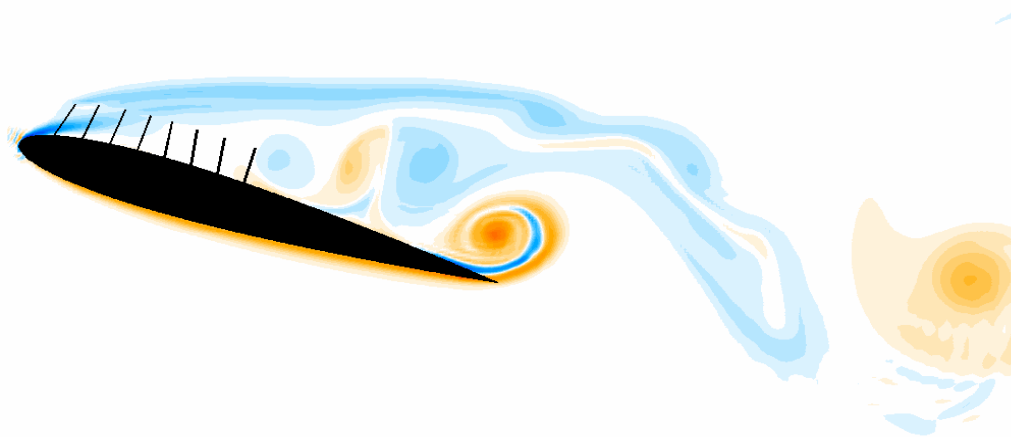
Control elements within 0.06 chord and 0.45 chord

$$T_{structure} \approx T_{fluid} = 1.53$$

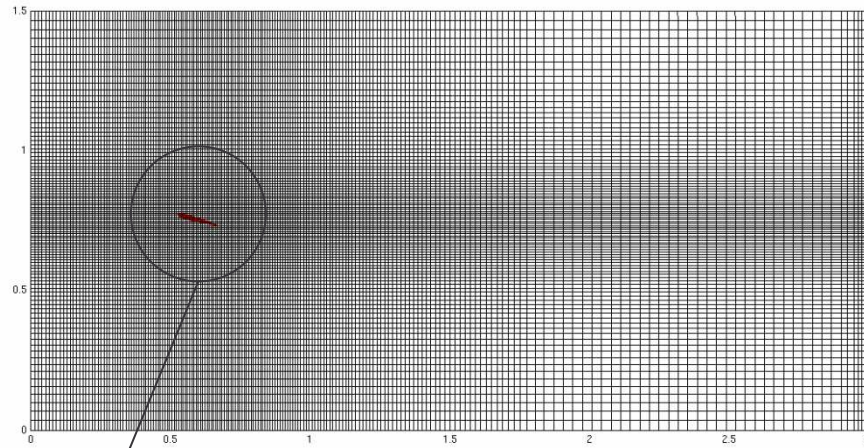


$$\alpha = 18^\circ$$

$$\rho_{feathers} = 890 \text{ Kg/m}^3 \text{ (keratin)}$$

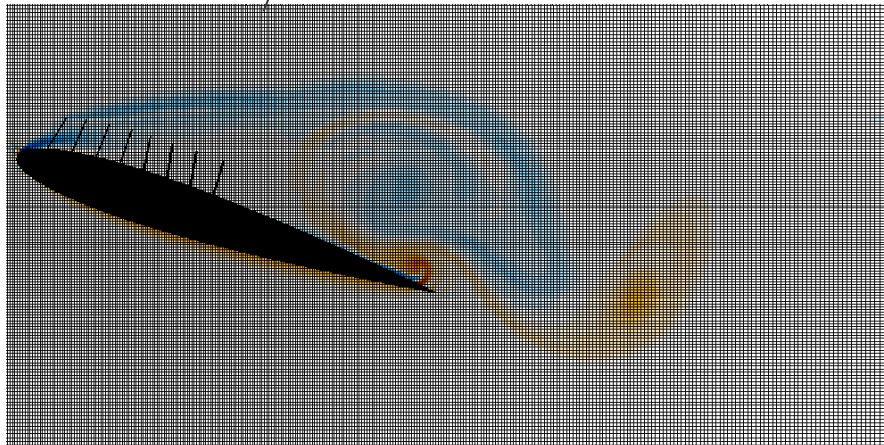


$$\alpha = 18^\circ$$



40 chords x 20 chords

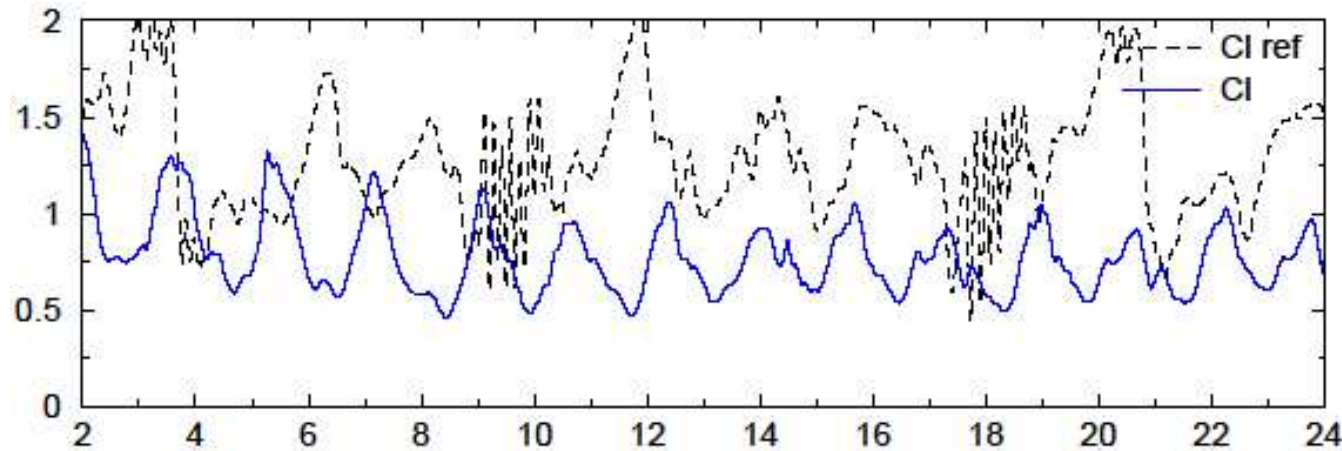
grid: 1200 x 600



Drag reduction by about 15%

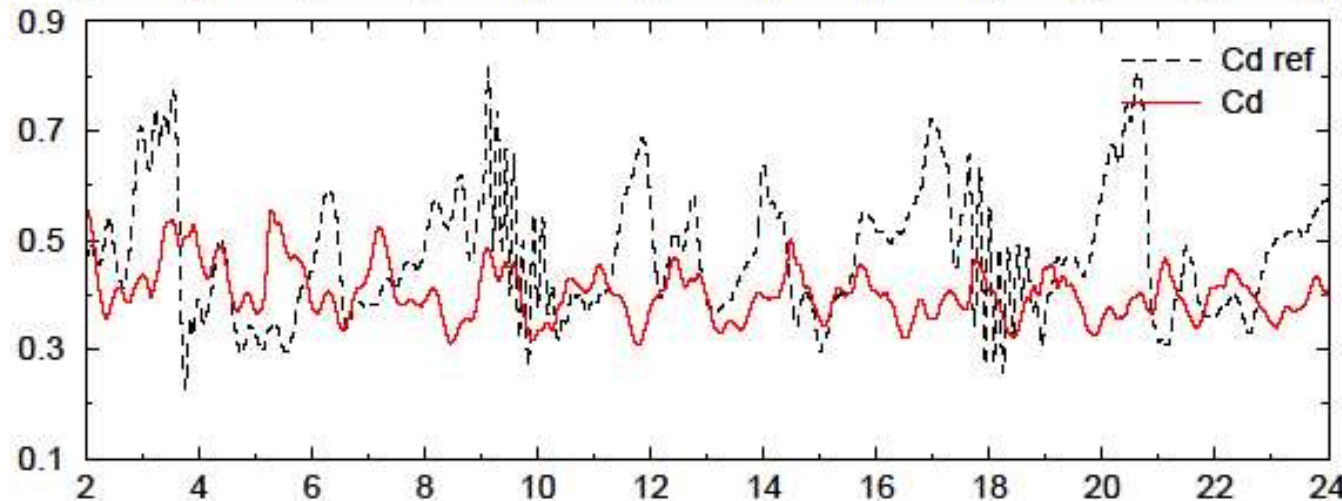
Lift reduction by about 40%

About 60% reduction in the amplitude of the oscillations



$$\langle C_l \rangle = 1.27$$

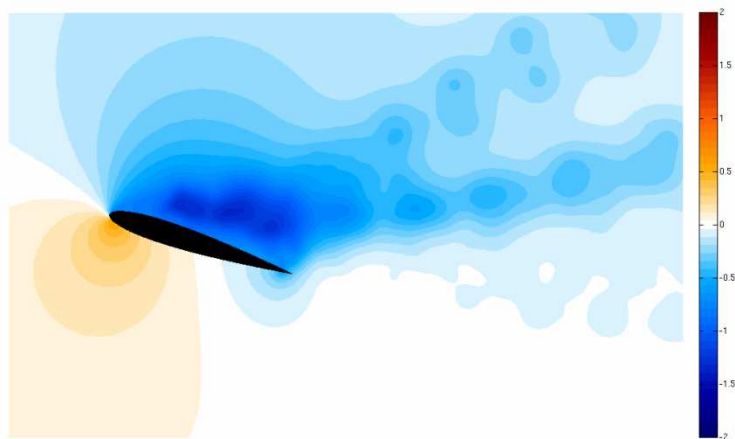
$$\langle C_l \rangle = 0.76$$



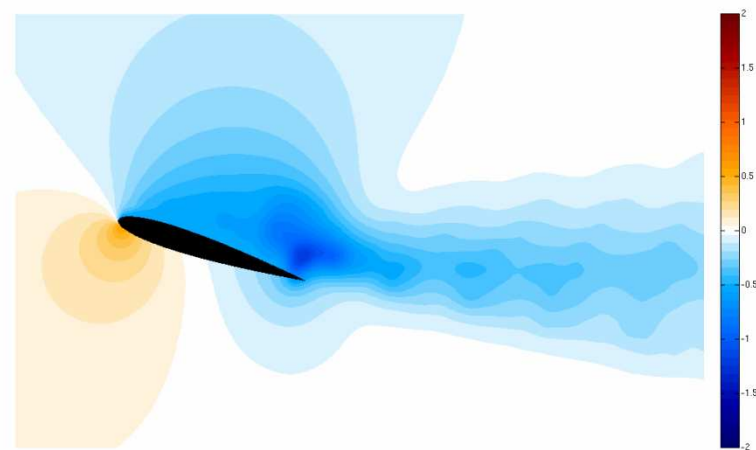
$$\langle C_d \rangle = 0.475$$

$$\langle C_d \rangle = 0.402$$

Mean pressure field



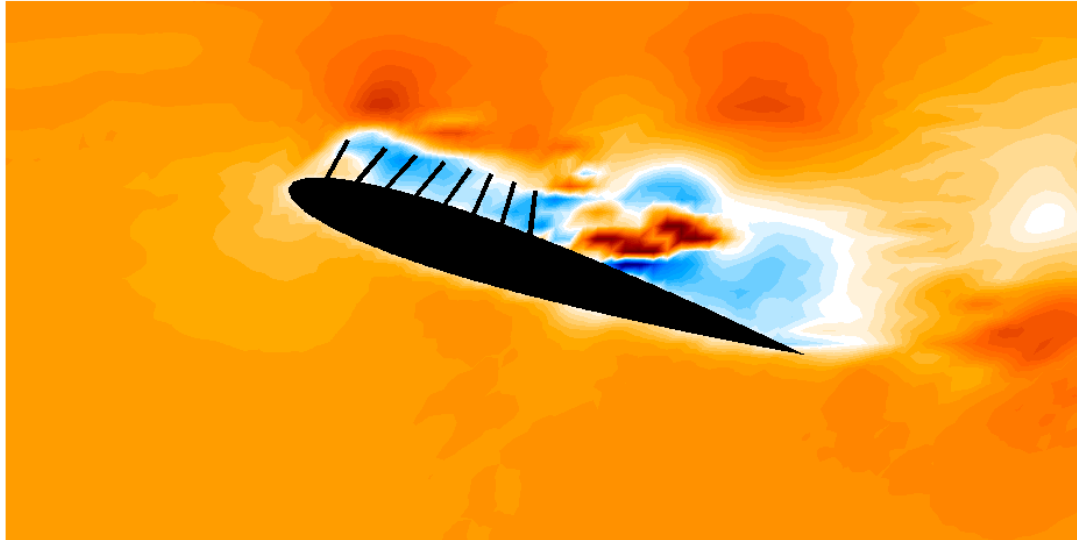
no control



with control

Issues left:

- extend the parametric search
- link the properties of the optimised structures to those of a suitable material
- wind tunnel/water channel tests (Ch. Brücker, U. Freiberg)



Engineering perspectives

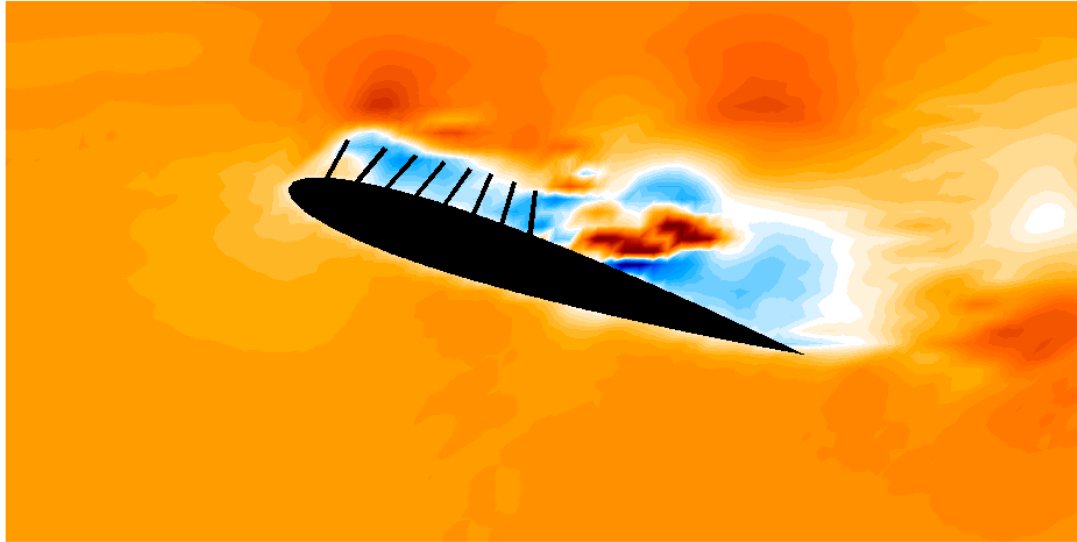
MAV/UAV

Car, trucks and trains – underwater vehicles

Wind turbines

Hydraulic machines (cavitation)

Sound mitigation



Paleontological perspective

Dinosaur ancestry of birds: could the “feathery” dinosaurs discovered in the Liaoning province fly?

Fluid mechanics could say something on the role of those protofeathers ...



