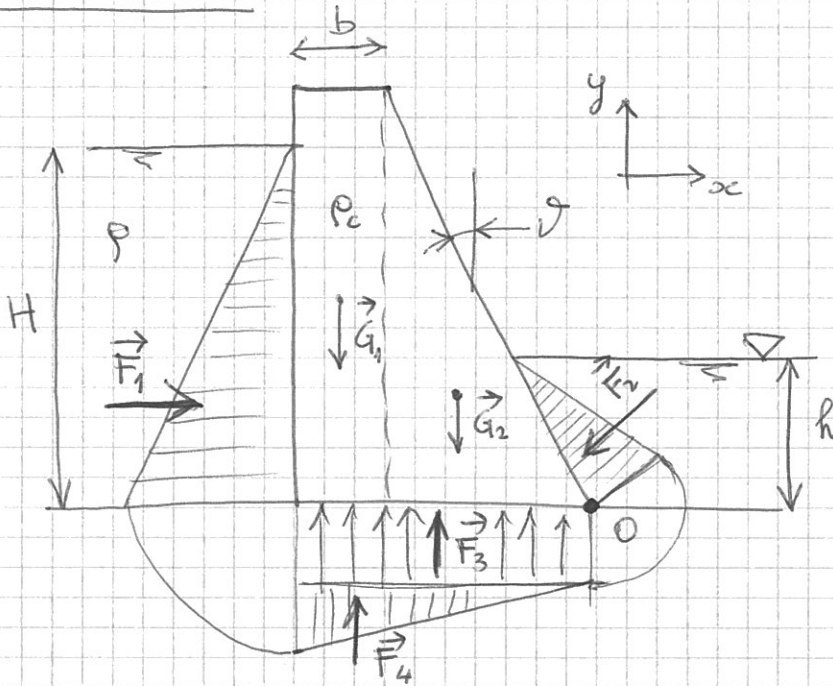


ESERCIZIO 1



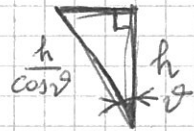
$$\begin{aligned} \vec{F}_1 &= (F_1, 0) \\ \vec{F}_2 &= (F_{2x}, F_{2y}) \\ \vec{F}_3 &= (0, F_3) \\ \vec{F}_4 &= (0, F_4) \\ \vec{G}_1 &= (0, -G_1) \\ \vec{G}_2 &= (0, -G_2) \end{aligned}$$

$$F_1 = \gamma \frac{H^2}{2}$$

$$b_1 = \frac{H}{3} \quad (\text{bracci rispetto al polo } O)$$

$$|F_2| = \gamma \frac{R^2}{\cos^2 \varphi} \frac{1}{2}$$

$$b_2 = \frac{h}{3 \cos^2 \varphi}$$



$$\tan \varphi = \frac{B-b}{H+\Delta}$$

$$F_3 = \gamma h B$$

$$b_3 = \frac{B}{2}$$

$$F_4 = \gamma \frac{(H-h)B}{2}$$

$$b_4 = \frac{2}{3} B$$

$$G_1 = \rho_c g b (H+\Delta) \quad b_{G_1} = B - \frac{b}{2}$$

$$G_2 = \rho_c g (B-b)(H+\Delta) \quad b_{G_2} = (B-b) \frac{2}{3}$$

La diga sarà stabile al ribaltamento attorno al polo O se:

$$G_1 b_{G_1} + G_2 b_{G_2} + |F_2| b_2 \geq F_1 b_1 + F_3 b_3 + F_4 b_4$$

Momento stabilizzante

Momento destabilizzante

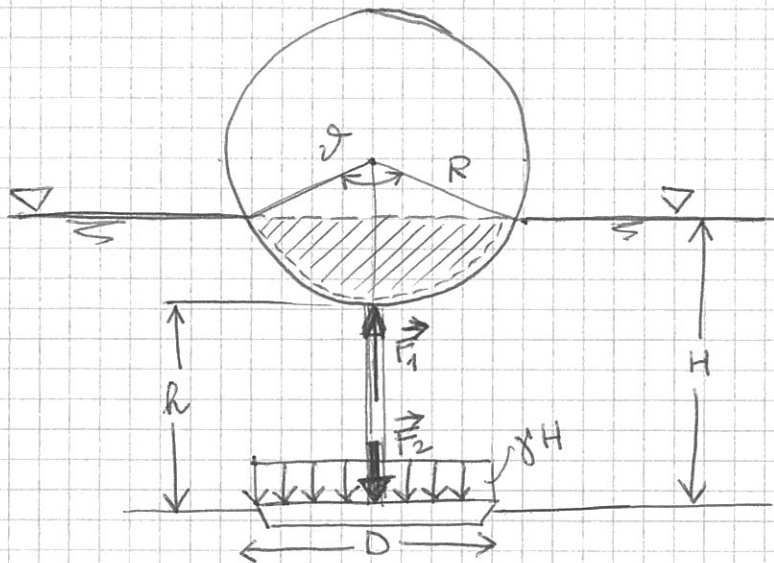
Risulta

$$M_{STAB} = 14408,9 + 17077,2 + 110,4 = 31596,6 \text{ kN} \cdot \text{m/m}$$

$$M_{DEST} = 44145 + 1962 + 8502 = 54609 \text{ kN} \cdot \text{m/m}$$

$$\Rightarrow M_{DEST} > M_{STAB} \Rightarrow \text{INSTABILE}$$

ESERCIZIO 2



Per valutare la spinta esercitata dal fluido sul cilindro si consideri il volume tratteggiato. Per l'equilibrio risulta:

$$\vec{G} + \vec{F}_1 = 0$$

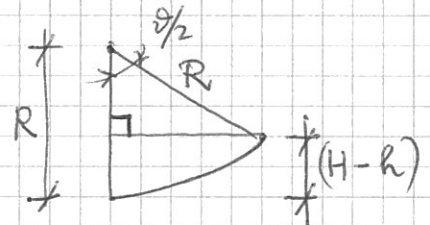
$$\vec{G} = (0, -G)$$

$$\vec{F}_1 = (0, F_1)$$

$$F_1 - G = 0$$

$$F_1 = G = \gamma \left[R^2 \frac{\vartheta}{2} - R^2 \frac{\cos(\vartheta/2) \sin(\vartheta/2)}{2} \right] L = \gamma R^2 \left[\frac{\vartheta}{2} - \frac{\sin \vartheta}{2} \right] L$$

$$\cos(\vartheta/2) = \frac{R - (H - h)}{R} = 1 - \frac{(H - h)}{R}$$



Le spinte esercitate sulle valvole risultano:

$$\vec{F}_2 = (0, -F_2)$$

$$F_2 = \gamma H \frac{\pi D^2}{4}$$

Il cilindro si solleverà quando:

$$F_1 - F_2 \geq P$$

Procedendo per tentativi si trova: $H = 2,34 \text{ m}$

ESERCIZIO 3

a) Perdite trascurabili

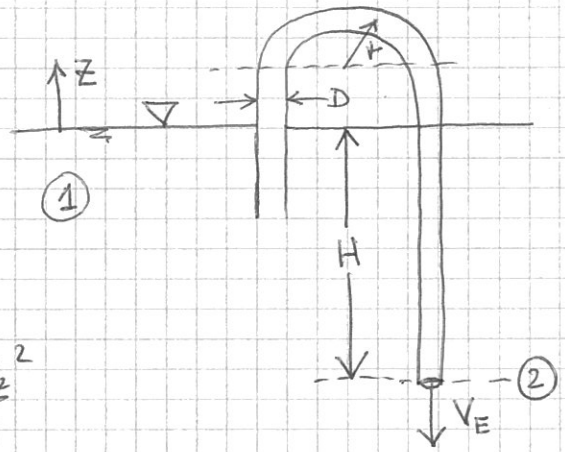
$$H_1 = H_2$$

$$H_1 = z_1 + \frac{p_1}{\rho} + \frac{v_1^2}{2g} = 0$$

$$H_2 = z_2 + \frac{p_2}{\rho} + \frac{v_2^2}{2g} = -H + \frac{v_2^2}{2g}$$

$$\Rightarrow -H + \frac{v_2^2}{2g} = 0 \Rightarrow v_2 = \sqrt{2gH} = 4,43 \text{ m/s}$$

$$Q = \Omega V = \frac{\pi D^2}{4} v = 0,00139 \text{ m}^3/\text{s} = 1,39 \text{ l/s}$$



b) Con dissipazioni

$$H_1 - \frac{v^2}{2g} \left(\xi_{\text{imbocco}} + \xi_{\text{curva}} + \lambda L/D \right) = H_2$$

$$\xi_{\text{imbocco}} \approx 1$$

$$\xi_{\text{curva}} \rightarrow \frac{r}{D} = \frac{0,25}{0,02} = 12,5 \quad \theta = 180^\circ \rightarrow \xi \approx 0,4$$

$$\begin{cases} \text{Re} = \frac{vD}{\nu} = \\ \epsilon = \frac{\epsilon_s}{D} = \end{cases} \rightarrow \lambda \text{ from Moody's diagram}$$

$$H_1 = 0$$

$$H_2 = -H + \frac{v^2}{2g}$$

$$\Rightarrow H_2 - H_1 + \frac{v^2}{2g} \left(\xi_{\text{imb}} + \xi_{\text{curv}} + \frac{\lambda L}{D} \right) = 0$$

$$-H + \frac{v^2}{2g} \left(\xi_{\text{imb}} + \xi_{\text{curv}} + \frac{\lambda L}{D} + 1 \right) = 0$$

Procedendo per tentativi si trova:

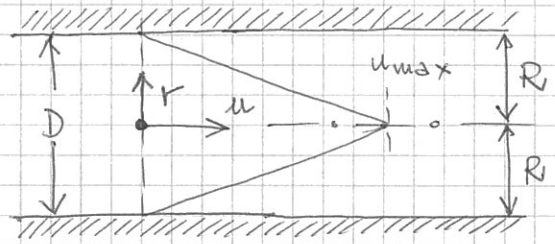
$v [\text{m/s}]$	$Q [\text{l/s}]$	Re	λ	Bilancio Energetico $\stackrel{?}{=} 0$	OUTPUT
4,43	1,392	$2,18 \cdot 10^5$	0,0187	4,0	$v \downarrow$
1,0	0,314	$4,92 \cdot 10^4$	0,0226	-0,717	$v \uparrow$
2,0	0,628	$9,85 \cdot 10^4$	0,0204	0,067	$v \downarrow$
1,93	0,607	$9,52 \cdot 10^4$	0,0204	-0,0001	OK

OK

③

ESERCIZIO 4

a) $u(r) = \left(1 - \frac{r}{R}\right) u_{\max}$



$$Q = \int_{\Omega} (\vec{u} \cdot \vec{n}) d\Omega = \int_0^{2\pi} \int_0^R u(r) r dr d\vartheta =$$

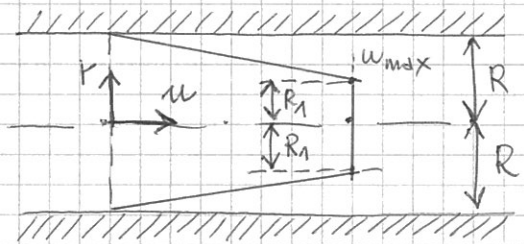
$$= u_{\max} 2\pi \int_0^R \left(1 - \frac{r}{R}\right) r dr = u_{\max} 2\pi \left[\frac{r^2}{2} - \frac{r^3}{3R} \right]_0^R =$$

$$= u_{\max} 2\pi \left(\frac{R^2}{2} - \frac{R^2}{3} \right) = u_{\max} 2\pi \frac{R^2}{6} = u_{\max} \frac{\pi R^2}{3} = 75,4 \cdot 10^{-3} \text{ m}^3/\text{s}$$

$$Q_m = \rho Q = 75,4 \text{ kg/s}$$

$$U = \frac{Q}{\Omega} = \frac{Q}{\pi R^2} = 6,67 \text{ m/s}$$

b)
$$\begin{cases} u(r) = \left[1 - \frac{(r-R_1)}{(R-R_1)}\right] u_{\max} & R_1 < r < R \\ u(r) = u_{\max} & r < R_1 \end{cases}$$



$$Q = u_{\max} \pi R_1^2 + \int_0^{2\pi} \int_{R_1}^R u(r) r dr d\vartheta = u_{\max} \pi R_1^2 + 2\pi u_{\max} \left[\frac{r^2}{2} - \frac{r^3}{3(R-R_1)} + \frac{r^2 R_1}{2(R-R_1)} \right]_{R_1}^R$$

$$= u_{\max} \pi R_1^2 + 2\pi u_{\max} \left[\frac{R^2 - R_1^2}{2} - \frac{R^3 - R_1^3}{3(R-R_1)} + \frac{(R^2 - R_1^2) R_1}{2(R-R_1)} \right] =$$

$$= u_{\max} \pi R_1^2 + 2\pi u_{\max} \left[\frac{R^2 - R_1^2}{2} - \frac{R^2 + RR_1 + R_1^2}{3} + \frac{(R + R_1) R_1}{2} \right] =$$

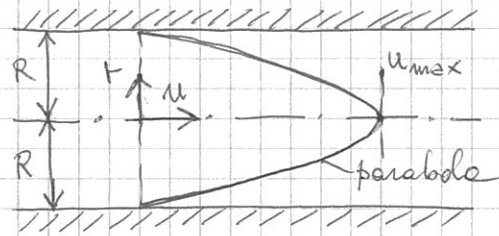
$$= u_{\max} \pi R_1^2 + 2\pi u_{\max} \left[\frac{R^2}{6} + \frac{RR_1}{6} - \frac{R_1^2}{3} \right] \quad \leftarrow R_1 = R/2$$

$$= u_{\max} \pi R^2/4 + 2\pi u_{\max} \left[\frac{R^2}{6} + \frac{R^2}{12} - \frac{R^2}{12} \right] = u_{\max} \frac{7\pi R^2}{12} = 131,9 \cdot 10^{-3} \text{ m}^3/\text{s}$$

$$Q_m = \rho Q = 131,9 \text{ kg/s}$$

$$U = \frac{Q}{\Omega} = \frac{Q}{\pi R^2} = 11,67 \text{ m/s}$$

$$c) \quad u(r) = \left[1 - \left(\frac{r}{R} \right)^2 \right] u_{\max}$$



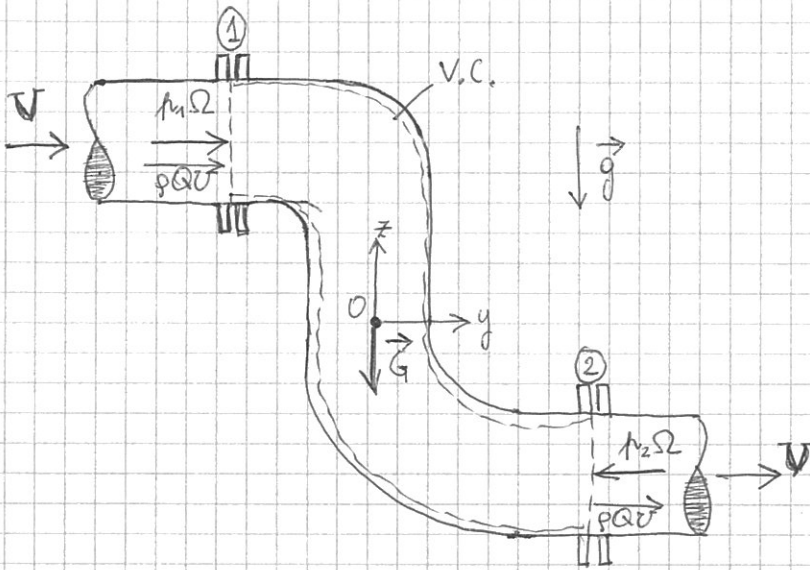
$$Q = \int_0^{2\pi} \int_0^R u(r) r dr d\vartheta = 2\pi u_{\max} \int_0^R \left[1 - \left(\frac{r}{R} \right)^2 \right] r dr =$$

$$= 2\pi u_{\max} \left[\frac{r^2}{2} - \frac{r^4}{4R^2} \right]_0^R = 2\pi u_{\max} \left(\frac{R^2}{2} - \frac{R^2}{4} \right) = u_{\max} \frac{\pi R^2}{2} = 113 \cdot 10^{-3} \frac{\text{m}^3}{\text{s}}$$

$$Q_m = \rho Q = 113,1 \frac{\text{kg}}{\text{s}}$$

$$v = \frac{Q}{S} = \frac{Q}{\pi R^2} = 10 \frac{\text{m}}{\text{s}}$$

ESERCIZIO 5



Moto staz. $\Rightarrow Q = \text{cost}$

$$Q = \Omega U =$$

$$\Omega = \text{cost} \Rightarrow U = \text{cost} = 3 \text{ m/s}$$

$$p_2 = p_1 - 50 = 225 \text{ kPa}$$

$$\vec{G} + \vec{\Pi} = \vec{I} + \vec{M}_u - \vec{M}_i$$

$$\vec{G} = (0, -G)$$

$$\vec{M}_u = (p_1 Q U, 0)$$

$$\vec{I} = 0$$

moto staz.

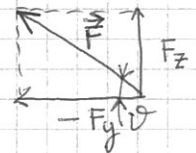
$$\vec{\Pi} = (p_1 \Omega - p_2 \Omega + F_y, F_z)$$

$$\vec{M}_i = (p_2 Q U, 0)$$

$$\vec{F} = (F_y, F_z) \quad \text{forze esercitate dalla condotta sul fluido}$$

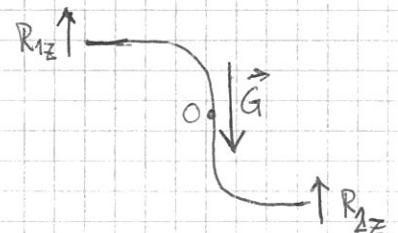
$$\begin{cases} y \\ z \end{cases} \begin{cases} (p_1 - p_2) \Omega + F_y = p_1 Q U - p_2 Q U = 0 \\ -G + F_z = 0 \end{cases} \begin{cases} F_y = -(p_1 - p_2) \Omega < 0 \\ F_z = G > 0 \end{cases}$$

$$\text{tg} \vartheta = \frac{F_z}{-F_y} = \frac{G}{(p_1 - p_2) \Omega}$$



La componente verticale F_z delle forze esercitate dalla condotta sul fluido sarà ripartita equamente tra le due flange (come suggerito nel testo). Risulta dunque:

$$R_{1z} = R_{2z} = \frac{F_z}{2} = \frac{G}{2}$$



Per calcolare le componenti orizzontali delle forze esercitate dalle flange bisogna applicare il principio del momento delle quantità di moto ($\sum M^+$):

(polo O)

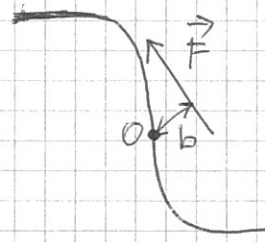
$$p_1 \Omega \frac{L}{2} + p_2 \Omega \frac{L}{2} - b |F| = -p_1 Q U \frac{L}{2} - p_2 Q U \frac{L}{2}$$

dove b è il braccio delle forze \vec{F} esercitate dalle condotte nel fluido

Il momento esercitato dalle condotte sarà dunque:

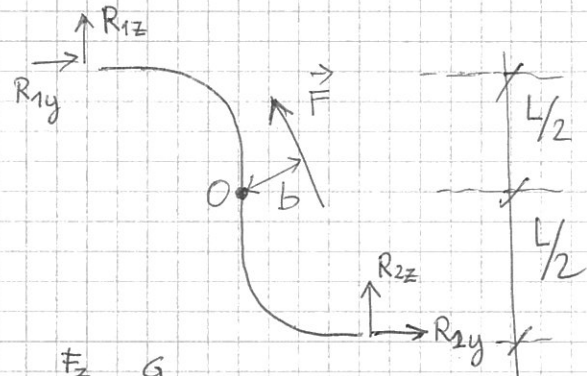
$$b|F| = (\rho_1 + \rho_2)\Omega \frac{L}{2} + \rho Q U L$$

e sarà uguale e contrario rispetto a quello generato dal fluido sulle condotte.



Per l'equilibrio delle condotte dovrà dunque risultare:

$$\begin{cases} \text{[y]} & R_{1y} + R_{2y} = F_y \\ \text{[z]} & R_{1z} + R_{2z} = F_z \\ \text{Mom.} & R_{1y} \frac{L}{2} - R_{2y} \frac{L}{2} = -b|F| \end{cases} \xrightarrow{\text{dip } R_{1z} = R_{2z}} R_{1z} = R_{2z} = \frac{F_z}{2} = \frac{G}{2}$$



Dalla eq. ne lungo y : $R_{1y} = F_y - R_{2y} = -(\rho_1 - \rho_2)\Omega - R_{2y}$
Sostituendo nell'eq. ne dei momenti:

$$-(\rho_1 - \rho_2)\Omega \frac{L}{2} - R_{2y} \frac{L}{2} - R_{2z} \frac{L}{2} = -(\rho_1 + \rho_2)\Omega \frac{L}{2} - \rho Q U L$$

$$\rightarrow R_{2y} = \rho_2 \Omega + \rho Q U \vec{e}_z$$

$$R_{1y} = -(\rho_1 - \rho_2)\Omega - \rho_2 \Omega - \rho Q U \vec{e}_z = -\rho_1 \Omega - \rho Q U \vec{e}_z$$

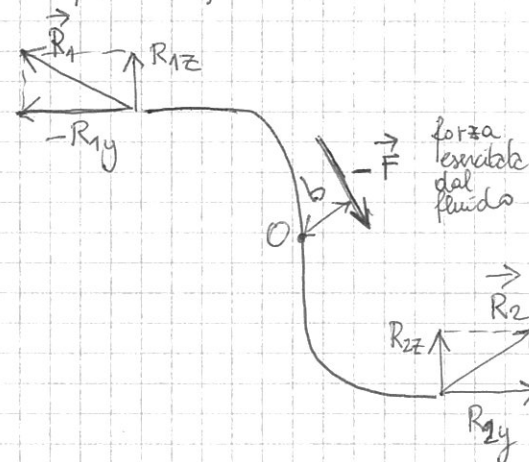
Utilizzando i valori numerici:

$$G = \gamma \frac{\pi D^2}{4} (L + 2H) = 0,770 \text{ kN}$$

$$R_{1z} = R_{2z} = \frac{G}{2} = 0,385 \text{ kN}$$

$$R_{1y} = -\rho_1 \Omega - \rho Q U \vec{e}_z = -2,231 \text{ kN}$$

$$R_{2y} = \rho_2 \Omega + \rho Q U \vec{e}_z = 1,838 \text{ kN}$$



ESERCIZIO 6

$$\tau = \mu \left. \frac{du}{dy} \right|_{y=0}$$

$$\frac{du}{dy} = u_{\max} \left[\frac{3}{2} \frac{1}{h} - \frac{3}{2} \left(\frac{y}{h} \right)^2 \frac{1}{h^2} \right]$$

$$\left. \frac{du}{dy} \right|_{y=0} = u_{\max} \frac{3}{2h}$$

$$\Rightarrow \tau = \mu \left. \frac{du}{dy} \right|_{y=0} = \mu u_{\max} \frac{3}{2h}$$

$$\Rightarrow h = \frac{\mu u_{\max}}{\tau} \frac{3}{2} = \frac{1.5 \cdot 4 \cdot 10^{-2}}{20} \cdot \frac{3}{2} = 4,5 \cdot 10^{-3} \text{ m}$$

ESERCIZIO 7

Similitudine di Froude $\rightarrow F_{r_m} = F_{r_p}$

$$F_{r_m} = \frac{U_m}{\sqrt{g L_m}} = \frac{U_p}{\sqrt{g L_p}} = F_{r_p}$$

$$\Rightarrow U_p = U_m \sqrt{\frac{L_p}{L_m}} = U_m \sqrt{20} = 3,6 \sqrt{20} = 16,1 \text{ m/s}$$

$$C_{D_m} = \frac{F_{D_m}}{\rho_m U_m^2 D_m^2} = \frac{F_{D_p}}{\rho_p U_p^2 D_p^2} = C_{D_p}$$

$$\Rightarrow F_{D_p} = F_{D_m} \left(\frac{\rho_p}{\rho_m} \right) \left(\frac{U_p}{U_m} \right)^2 \left(\frac{D_p}{D_m} \right)^2 = F_{D_m} \left(\frac{\rho_p}{\rho_m} \right) \left(\frac{L_p}{L_m} \right)^3 \stackrel{\text{hp } \rho_p = \rho_m}{=} F_{D_m} \left(\frac{L_p}{L_m} \right)^3 =$$
$$= 12,2 \cdot 20^3 = 97,6 \text{ kN}$$

ESERCIZIO 8

a) $F_M + A = P$

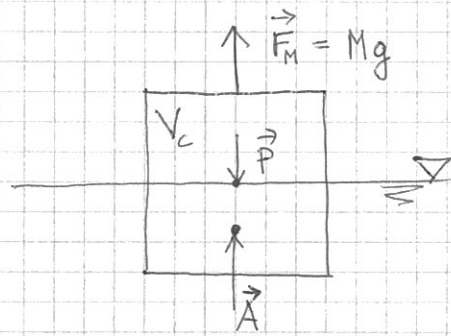
$F_M = Mg$

$A = \rho g \frac{V_c}{2}$

$P = \rho_s g V_c$

da cui $Mg + \rho g \frac{V_c}{2} = \rho_s g V_c$

$\rho_s = \frac{M}{V_c} + \frac{\rho}{2} = \frac{300}{1} + \frac{1000}{2} = 800 \text{ kg/m}^3$



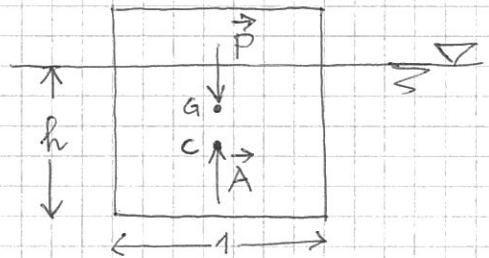
b) $\rho_s < \rho \rightarrow$ galleggia

Calcolo dell'affondamento h :

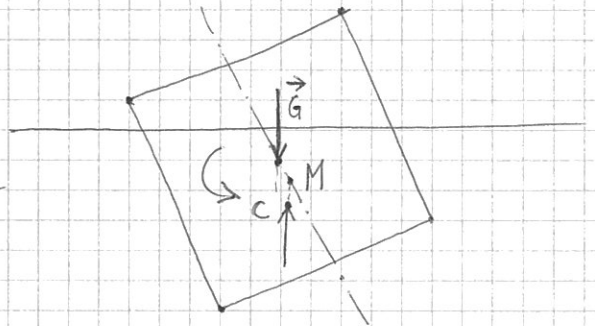
$|P| = |A|$

$\rho_s g V_c = \rho g h \cdot 1$

$\Rightarrow h = V_c \frac{\rho_s}{\rho} = 1 \cdot \frac{800}{1000} = 0,8 \text{ m}$



Presumibilmente il cubo è in equilibrio instabile perché con una piccola oscillazione si realizza una coppia destabilizzante.



c) $|P| = |A| + F_M$

$\rho_c V_c g = \rho V_c g + Mg$

$\rho_c V_c g = \rho V_c g + Mg$

$\Rightarrow \rho_c = \rho + \frac{M}{V_c} = 1000 + \frac{300}{1} = 1300 \text{ kg/m}^3$

